

**Control Systems**  
**05/06/2018**

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**Exercise 1** Denoting  $L(s) = G(s)P(s)$ , one has

$$y(s) = W(s)v(s), \quad e(s) = W_e(s)v(s)$$

with  $W(s) = \frac{L(s)}{1+L(s)}$  and  $W_e(s) = \frac{1}{1+L(s)}$ . As usual, we shall split the controller in two loops; namely,  $G(s) = G_2(s)G_1(s)$  with  $G_1(s)$  designed for satisfying steady-state specifications (i.e., (ii)) whereas the outer loop  $G_2(s)$  is defined for transient and stability requirements (i.e., (iii) and (i)).

(ii) Set for the time being  $G_2(s) = 1$ . As  $e_{ss}(t) = W_e(0)t + \frac{\partial W_e}{\partial s}(0)$ , when  $v(t) = t$ , one needs  $W_e(0) = 0$  and  $\frac{\partial W_e}{\partial s}(0) = 0$ . Accordingly, two integrating actions are needed. As the plant itself already possesses a pole at  $s = 0$ , we set the inner control loop  $G_1(s) = \frac{1}{s}$ .

(iii) By inspecting the Bode Plots of

$$L_1(s) = G_1(s)P(s) = \frac{1}{s^2(s-1)} \quad (1)$$

reported in Figure 1, one notices that the outer loop control action  $G_2(s)$  needs to be chosen so to

1. increase the value of the phase at  $\omega_t^* = 3$  rad/s as so that  $m_\varphi^* = 180^\circ + \angle G_2(j\omega_t^*) + \angle L_1(j\omega_t^*) \geq 30^\circ$  with  $\angle L_1(j\omega_t^*) = -360^\circ + 71.57^\circ$  so implying  $\angle G_2(j\omega_t^*) \geq 138.43^\circ$ .
2. decrease the magnitude at  $\omega_t^* = 3$  rad/s so to guarantee  $|G_2(j\omega_t^*)|_{dB} + |L_1(j\omega_t^*)|_{dB} = 0$  with  $|L_1(j\omega_t^*)|_{dB} \approx -29.08$

Accordingly, as no bound is apriori set over the gain of  $G_2(s)$  (that is  $G_2(0)$ ), we shall design the outer loop as composed of anticipating actions aimed at increasing the phase at  $\omega_t^* = 3$  rad/s and a gain to make  $\omega_t^* = 3$  rad/s the new cross-over frequency.

Thus, the structure we propose for  $G_2(s)$  is the following one

$$G_2(s) = kG_a(s), \quad k > 0$$

with  $G_a(s) = G_a^1(s)G_a^2(s)$ . In particular, we introduce 3 anticipating actions of the form

$$G_a^1(s) = \left( \frac{1 + \tau_a^1 s}{1 + \frac{\tau_a^1}{m_a^1} s} \right)^2, \quad G_a^2(s) = \frac{1 + \tau_a^2 s}{1 + \frac{\tau_a^2}{m_a^2} s}$$

with

- $G_a^1(s)$  composed of two identical anticipating functions acting at  $\omega_N^1 = 3$  rad/sec with  $m_a^1 = 16$  (that is at  $\tau_a^1 = 1$ ) so that  $\angle G_a^1(j\omega_t^*) \approx 121^\circ$  and  $|G_a^1(j\omega_t^*)|_{dB} \approx 20.08$ .
- $G_a^2(s)$  being one anticipating function with  $\omega_N^2 = 5$  rad/sec and  $m_a^2 = 3$  (that is  $\tau_a^2 = \frac{5}{3}$ ) so that  $\angle G_a^2(j\omega_t^*) \approx 20^\circ$  and  $|G_a^2(j\omega_t^*)|_{dB} \approx 8$ .

In this way, as  $k > 0$ ,  $m_\varphi^* = 32.57^\circ$  whereas  $k$  needs to be chosen so that  $|k|_{dB} + |G_a(j\omega_t^*)|_{dB} - 29.08 = 0$  with  $|G_a(j\omega_t^*)|_{dB} \approx 28.08$  so requiring  $k = 1.124$ . The bode

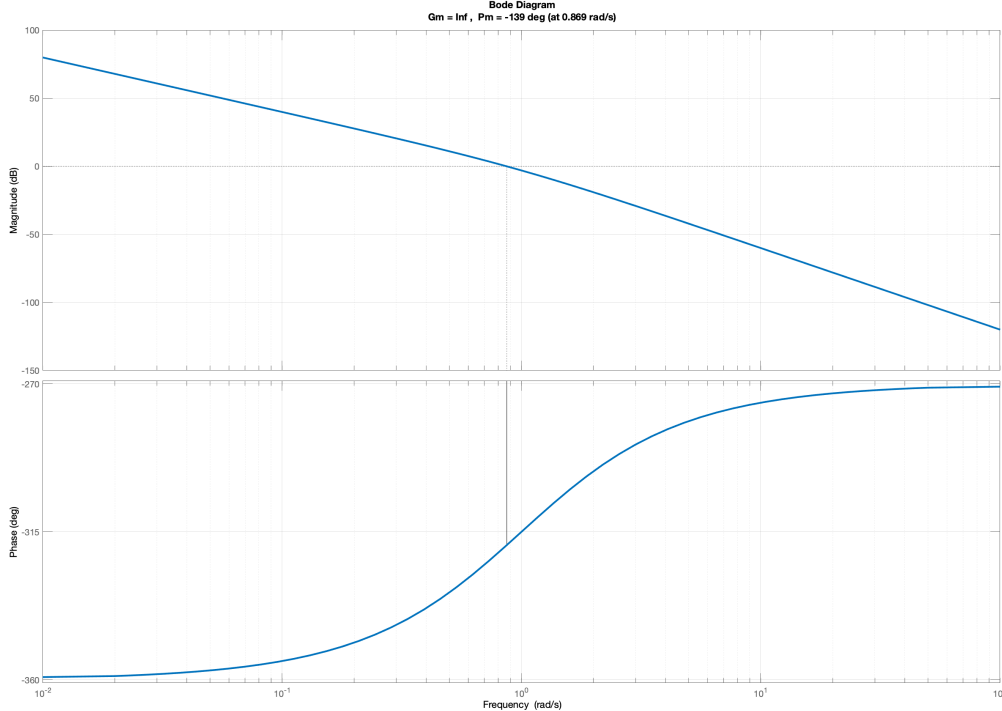


Figure 1: Bode plots of (1)

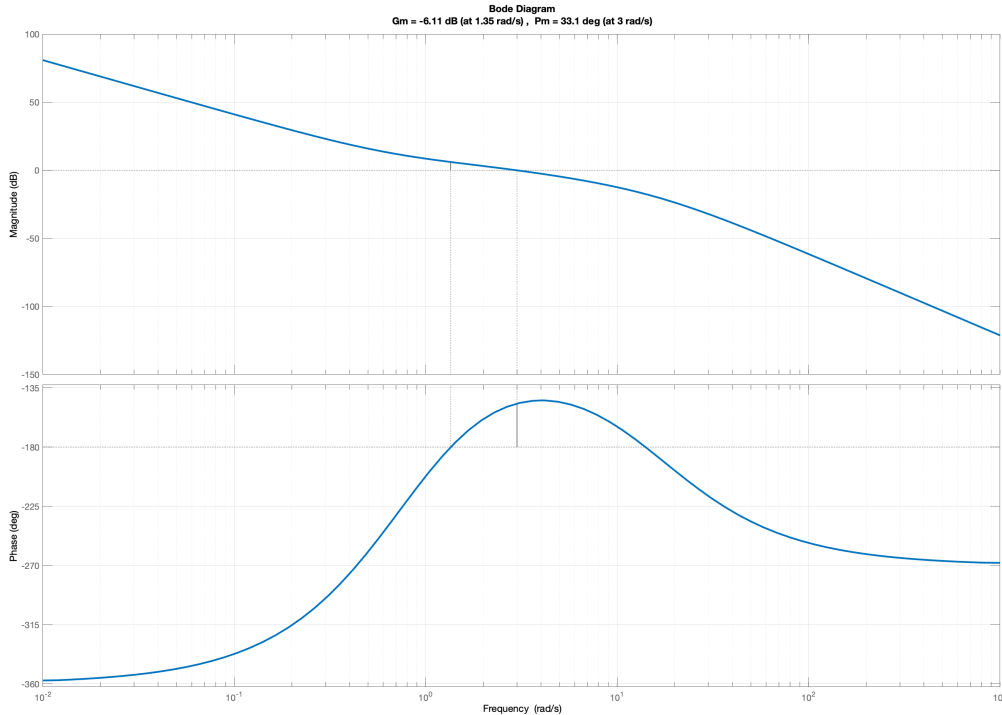


Figure 2: Bode plots of (2)

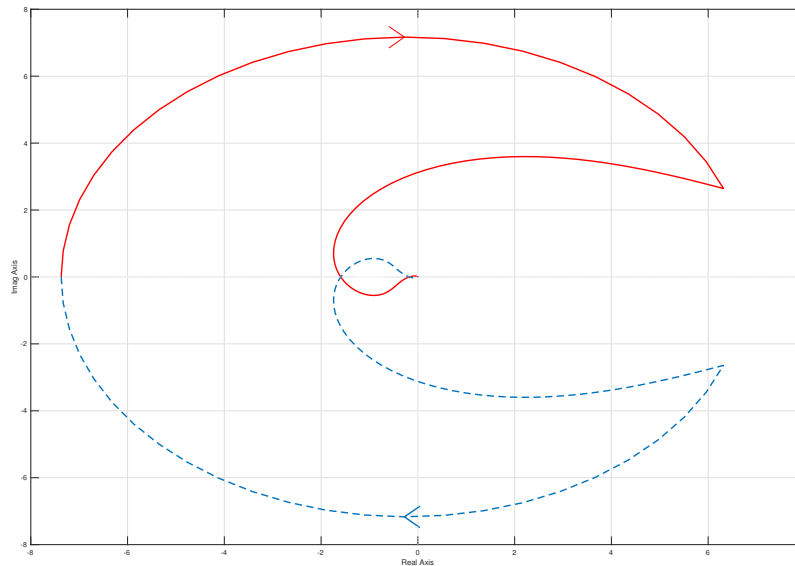


Figure 3: Nyquist plot of (2)

plots of

$$L(s) = G_2(s)G_1(s)P(s) = 1.124 \left( \frac{1+s}{1+\frac{1}{16s}} \right)^2 \frac{1+\frac{5}{3}s}{1+\frac{5}{9}s} \frac{1}{s^2(s-1)} \quad (2)$$

are reported in Figure 2.

- (i) The Nyquist plot of (2) is reported in Figure 3. The number of counter-clockwise encirclements of  $-1 + j0$  on behalf of the extended Nyquist plot of  $L(j\omega)$  is 1 that is also coincident with the open loop poles with positive real part. Thus, the system is asymptotically stable in closed loop.

**Exercise 2 a)** Denoting by  $n$  and  $m$  the number of poles and zeros of the transfer function, the relative degree of  $P(s)$  is given by  $r = n - m = 1$ . Accordingly, the root locus possesses two asymptotes centered at

$$s_0 = \frac{1+5}{2} = 3$$

that can be discarded. Introducing  $k \in \mathbb{R}$  and defining  $p(s, k) = (s^2+1)(s-1)+k(s+5)$  as the polynomial defining the closed-loop poles under  $G(s) = k$ , one gets that singularities are the solutions to

$$\begin{aligned} p(s, k) &= s^3 - s^2 + (k+1)s + 5k - 1 = 0 \\ \frac{\partial p(s, k)}{\partial s} &= 3s^2 - 2s + k + 1 = 0 \end{aligned}$$

By solving the equations above, it turns out that the negative locus possesses one sin-

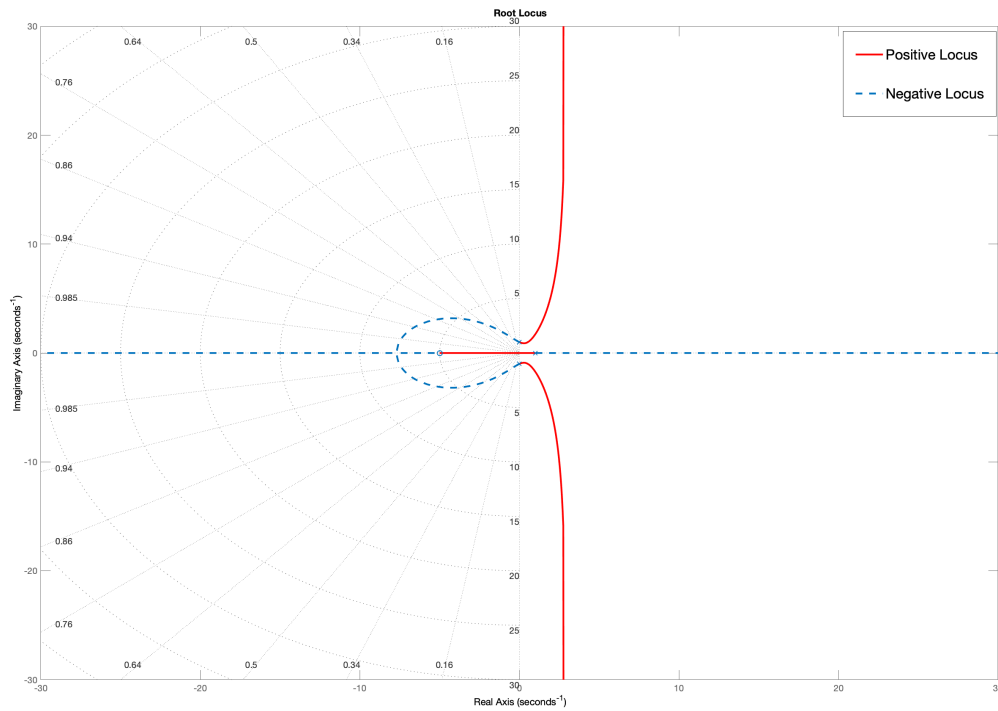


Figure 4: Root Locus of  $P(s) = \frac{s+5}{(s-1)(s^2+1)}$ .

gularity with multiplicity  $\mu = 2$  in correspondence of  $(s^*, k^*) \approx (-7.7, -194.27)$ . What is left to do is now is to quantify the number of intersection of the root locus with the imaginary axis. Those intersecting points correspond to values of  $k \in \mathbb{R}$  for which the Routh table of  $p(s, k) = s^3 + (k+9)s^2 + (4k+14)s + 4k - 24$  is not regular. Thus, by developing computations one gets

$$\begin{array}{l|ll} r^3 & 1 & k+1 \\ r^2 & -1 & 5k-1 \\ r^1 & -6k & \\ r^0 & 5k-1 & \end{array}$$

The Routh table is not regular for  $k = \frac{1}{5}$  and  $k = 0$  so implying that the positive locus intersects the imaginary axis in correspondence of  $k = \frac{1}{5}$  corresponding to the closed-loop pole  $s = 0$  and at  $k = 0$  corresponding to the open loop poles  $s = \pm j$ . The root locus is reported in Figure 4.

- b) From the above root locus and the Routh table it is evident that there exists no controller  $G(s) = k$  asymptotically stabilizing the feedback system.
- c) For ensuring zero-steady state error to constant inputs, the controller  $G(s)$  must possess a pole at  $s = 0$ . Thus, we set  $p_1 = 0$  and, for the sake of notations, we shall denote

hereinafter  $p = p_2$ . Thus, by denoting

$$L(s) = k \frac{(1 + z_1 s)(1 + z_2 s)(s + 5)}{s(s^2 + 1)(s - 1)(s + p)} = \hat{k} \frac{(s + \hat{z}_1)(s + \hat{z}_2)(s + 5)}{s(s^2 + 1)(s - 1)(s + p)}$$

one gets that a necessary condition for assigning the poles with real part smaller or equal than 3 is that the new center of the asymptotes satisfies

$$s'_0 = \frac{3 - p + \hat{z}_1 + \hat{z}_2}{2} < -3.$$

Accordingly,  $p$ ,  $\hat{z}_1$  and  $\hat{z}_2$  can be fixed as  $p = 25$ ,  $\hat{z}_1 = 3$  and  $\hat{z}_2 = 4$  so getting  $s'_0 = -6$  and implying  $z_1 = \frac{1}{3}$ ,  $z_2 = \frac{1}{4}$  and  $k = 12\hat{k}$ . At this point, one can set  $\hat{k} \in \mathbb{R}$  (or equivalently  $k \in \mathbb{R}$ ) by invoking the extended Routh criterion. Namely, one sets  $\hat{k}$  so to make the shifted closed-loop polynomial

$$\begin{aligned} p_L^*(s, \hat{k}) &= p_L(s - 3, \hat{k}) = (s - 3)(s^2 - 6s + 10)(s - 4)(s + 22) + \hat{k}s(s + 1)(s + 2) \\ &= s^5 + 9s^4 + (\hat{k} - 222)s^3 + (3\hat{k} + 1266)s^2 + (2\hat{k} - 3004)s + 2640 \end{aligned}$$

Hurwitz. By computing the Routh table

$$\begin{array}{l|llll} r^5 & 1 & \hat{k} - 222 & 2\hat{k} - 3004 & \\ r^4 & 3 & \hat{k} + 422 & 880 & \\ r^3 & \hat{k} - 544 & 3\hat{k} - 4946 & & \\ r^2 & \frac{\hat{k}^2 - 131\hat{k} - 214730}{\hat{k} - 544} & 880 & & \\ r^1 & \frac{(\hat{k}^3 - 2073\hat{k}^2 + 320392\hat{k} + 267210300)}{(\hat{k} - 544)^2} & & & \\ r^0 & 880 & & & \end{array}$$

one gets the specification satisfied for  $\hat{k} > 1815.4$ .

**Exercise 3 (i)** For computing the forced response, one needs to rewrite the input

$$u(t) = \begin{cases} 1 - e^{t-1} & \text{as } t \in [0, 1) \\ t - 1 & \text{as } t \in [1, 2) \\ 1 & \text{as } t \geq 2. \end{cases}$$

as the linear combination of elementary signals. Accordingly, one gets

$$u(t) = u_1(t) - e^{-1}u_2(t) - u_1(t - 1) + u_2(t - 1) + u_3(t - 1) - u_3(t - 2)$$

with

$$u_1(t) = 1_+, \quad u_2(t) = e_+^t, \quad u_3(t) = t_+.$$

Accordingly, as the system is time-invariant and linear, the output response can be computed as

$$y(t) = y_1(t) - e^{-1}y_2(t) - y_1(t - 1) + y_2(t - 1) + y_3(t - 1) - y_3(t - 2) \quad (3)$$

with

$$y_i(t) = \mathcal{L}^{-1}(P(s)u_i(s))[t], \quad u_i(s) = \mathcal{L}(u_i(t))[s], \quad i = 1, 2, 3.$$

In particular, one has

$$\begin{aligned} y_1(t) &= K\mathcal{L}^{-1}\left(\frac{1}{s(1+s)}\right)[t] = K\mathcal{L}^{-1}\left(\frac{1}{s}\right)[t] - K\mathcal{L}^{-1}\left(\frac{1}{1+s}\right)[t] = K(1_+ - e_+^{-t}) \\ y_2(t) &= K\mathcal{L}^{-1}\left(\frac{1}{(s-1)(s+1)}\right)[t] = \frac{K}{2}\mathcal{L}^{-1}\left(\frac{1}{s-1}\right)[t] - \frac{K}{2}\mathcal{L}^{-1}\left(\frac{1}{s+1}\right)[t] = \frac{K}{2}(e_+^t - e_+^{-t}) \\ y_3(t) &= K\mathcal{L}^{-1}\left(\frac{1}{s^2(s+1)}\right)[t] = K\mathcal{L}^{-1}\left(\frac{1}{s+1}\right)[t] - K\mathcal{L}^{-1}\left(\frac{1}{s}\right)[t] + \mathcal{L}^{-1}\left(\frac{1}{s^2}\right)[t] \\ &\quad K(e_+^{-t} - 1_+ + t_+). \end{aligned}$$

By substituting, after suitable time-shift, the above equalities in (3) one gets the result.

- (ii) The system has a well-define steady-state response as it is asymptotically stable (all poles, that we assume also as eigenvalues, are with negative real part). The steady-state response can be computed starting from (3) by neglecting all terms whose effect is vanishing in time. Accordingly, one gets

$$y_{ss}(t) = K.$$

- (iii) The settling time is defined as the time instant  $T_s > 0$  for which the output response remains within 5% of its steady-state values for all  $t \geq T_s$ . Accordingly, by defining the transient response as  $y_{\text{tran}}(t) = y(t) - y_{ss}(t)$  one gets that  $K$  needs to be chosen so that, for  $T_s \leq 10^{-3}$  and for all  $t \geq T_s$

$$|y_{\text{tran}}(t)| \leq 0.05|y_{ss}(t)|.$$

By rewriting  $y_{\text{tran}}(t) = K\bar{y}_{\text{tran}}(t)$  and  $y_{ss}(t) = K\bar{y}_{ss}(t)$  one gets that the above equality is independent upon  $K$  so that it is not possible to set the gain to decrease at will the settling time.