

Dottorato di Ricerca in Ingegneria dei Sistemi

Control of Nonholonomic Systems

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LECTURE OUTLINE

1. Introduction

- nonholonomic systems? among the others
- kinematic constraints
- integrability of kinematic constraints
- a control viewpoint
- dynamics vs. kinematics
- more general nonholonomic constraints

2. Modeling Examples

- wheeled mobile robots
 - * unicycle
 - * car-like robot
 - * N -trailer system
 - * other wheeled mobile robots
- space robots with planar structure
 - * two-body robot
 - * N -body robot

3. Tools from Differential Geometry

- Frobenius theorem
- integrability of Pfaffian constraints

4. Control Properties

- controllability of nonholonomic systems
- stabilizability of nonholonomic systems
- classification of nonholonomic distributions
- examples of classification

5. Nonholonomic Motion Planning

- chained forms
 - * steering with sinusoidal inputs
 - * steering with piecewise-constant inputs
 - * steering with polynomial inputs
 - * transformation into chained form
- WMRs in chained form
- unicycle simulation
- a general viewpoint: differential flatness

6. Feedback Control of Nonholonomic Systems

- basic problems
- asymptotic tracking
 - * control properties
 - * linear control design
 - * nonlinear control design
 - * dynamic feedback linearization
 - * experiments with SuperMario
- posture stabilization: a bird's eye view

7. Optimal Trajectories for WMRs

(by M. Vendittelli)

- minimum time problems
- application to WMRs
 - * extracting information from PMP
 - * type A trajectories
 - * type B trajectories

REFERENCES

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A Mathematical Introduction to Robotic Manipulation, CRC Press, 1994

relevant for Lectures 1–5

A. De Luca, G. Oriolo*

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relevant for Lectures 1–5

A. De Luca, G. Oriolo, C. Samson*

“Feedback control of a nonholonomic car-like robot”, in *Robot Motion Planning and Control* (J.-P. Laumond Ed.), Springer-Verlag, 1998

relevant for Lectures 4–6

G. Oriolo, A. De Luca, M. Vendittelli

“WMR control via dynamic feedback linearization: Design, implementation and experimental validation”*, *IEEE Transactions on Control System Technology*, vol. 10, no. 6, pp. 835–852, 2002.

relevant for Lectures 5–6

P. Souères, J.-D. Boissonnat†

“Optimal Trajectories for Nonholonomic Mobile Robots”, in *Robot Motion Planning and Control* (J.-P. Laumond Ed.), Springer-Verlag, 1998

relevant for Lecture 7

... and the references therein

* downloadable from

<http://www.dis.uniroma1.it/~labrob/people/oriolo/oriolo.html>

† downloadable from

<http://http://www.laas.fr/~jpl/book-toc.html>

INTRODUCTION

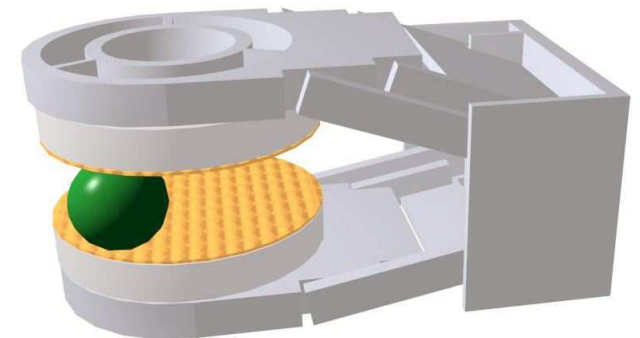
Nonholonomic systems? Among the others. . .



wheeled mobile robots (WMRs)

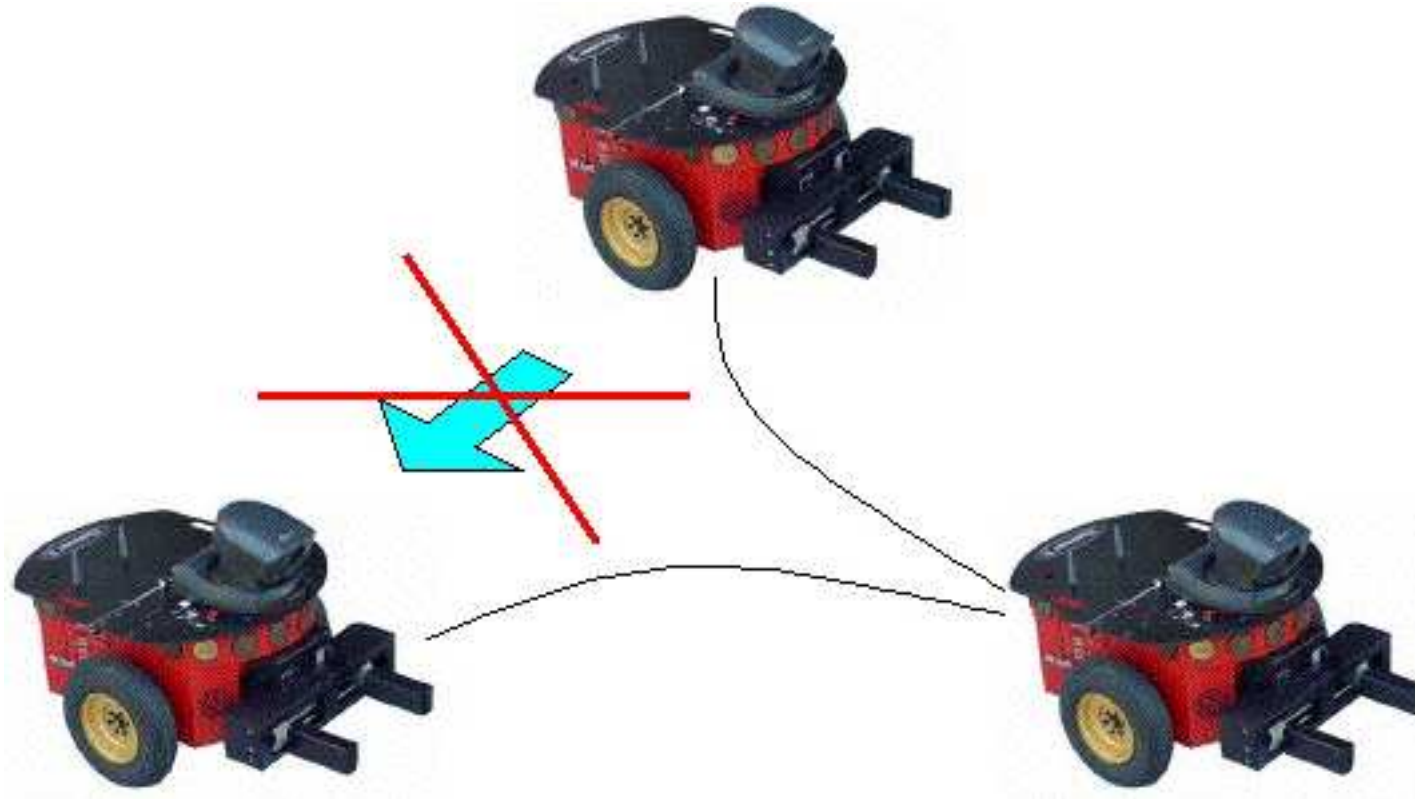


rolling manipulation



what is nonholonomy?

due to the presence of wheels, a WMR **cannot move sideways**



this is the **rolling without slipping** constraint, a special case of **nonholonomic** behavior

in general: a **nonholonomic** mechanical system **cannot move in arbitrary directions** in its configuration space

problems:

- our everyday experience indicates that WMRs are controllable, but can it be proven?
↔ we need a mathematical characterization of nonholonomy
- in any case, if the robot must move between two configurations, a **feasible** path is required (i.e., a motion that complies with the constraint)
↔ we need appropriate path planning techniques
- the feedback control problem is much more complicated, because:
 - ◇ a WMR is **underactuated**: less control inputs than generalized coordinates
 - ◇ a WMR is **not smoothly stabilizable** at a point↔ we need appropriate feedback control techniques

Kinematic Constraints

- the configuration of a mechanical system can be uniquely described by an n -dimensional vector of **generalized coordinates**

$$q = (q_1 \quad q_2 \quad \dots \quad q_n)^T$$

- the configuration space \mathcal{Q} is in general an n -dimensional smooth manifold, locally diffeomorphic to \mathbb{R}^n
- the **generalized velocity** at a generic point of a trajectory $q(t) \subset \mathcal{Q}$ is the tangent vector

$$\dot{q} = (\dot{q}_1 \quad \dot{q}_2 \quad \dots \quad \dot{q}_n)^T$$

- **geometric constraints** may exist or be imposed on the mechanical system

$$h_i(q) = 0 \quad i = 1, \dots, k$$

restricting the possible motions to an $(n - k)$ -dimensional submanifold

- a mechanical system may also be subject to a set of **kinematic constraints**, involving generalized coordinates and their derivatives; e.g., first-order kinematic constraints

$$a_i(q, \dot{q}) = 0 \quad i = 1, \dots, k$$

- in most cases, the constraints are **Pfaffian**

$$a_i^T(q)\dot{q} = 0 \quad i = 1, \dots, k \quad \text{or} \quad A^T(q)\dot{q} = 0$$

i.e., they are linear in the velocities

- kinematic constraints may be **integrable**, that is, there may exist k functions h_i such that

$$\frac{\partial h_i(q(t))}{\partial q} = a_i^T(q) \quad i = 1, \dots, k$$

in this case, the kinematic constraints are indeed geometric constraints

a set of Pfaffian constraints is called **holonomic** if it is integrable (a geometric limitation); otherwise, it is called **nonholonomic** (a kinematic limitation)

holonomic/nonholonomic constraints affect mobility in a **completely different** way:

for illustration, consider a single Pfaffian constraint

$$a^T(q)\dot{q} = 0$$

- if the constraint is **holonomic**, then it can be integrated as

$$h(q) = c$$

with $\frac{\partial h}{\partial q} = a^T(q)$ and c an integration constant



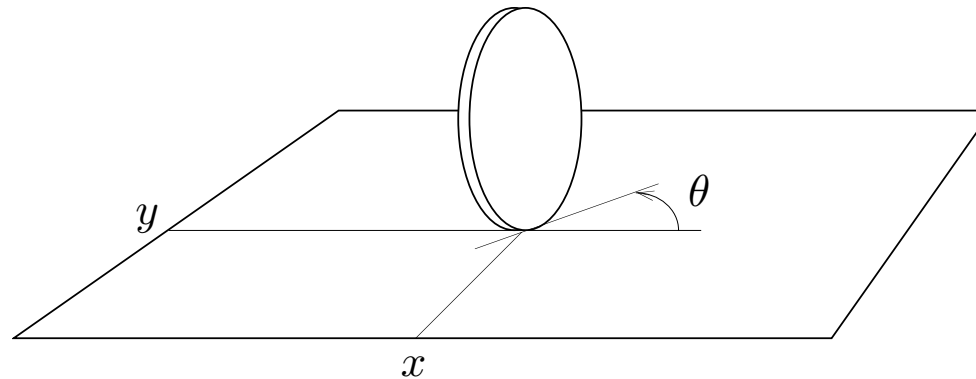
the motion of the system is confined to lie on a particular level surface (**leaf**) of h , depending on the initial condition through $c = h(q_0)$

- if the constraint is **nonholonomic**, then it cannot be integrated



although at each configuration the instantaneous motion (velocity) of the system is restricted to an $(n - 1)$ -dimensional space (the null space of the constraint matrix $a^T(q)$), **it is still possible to reach any configuration in \mathcal{Q}**

a canonical example of nonholonomy: the rolling disk

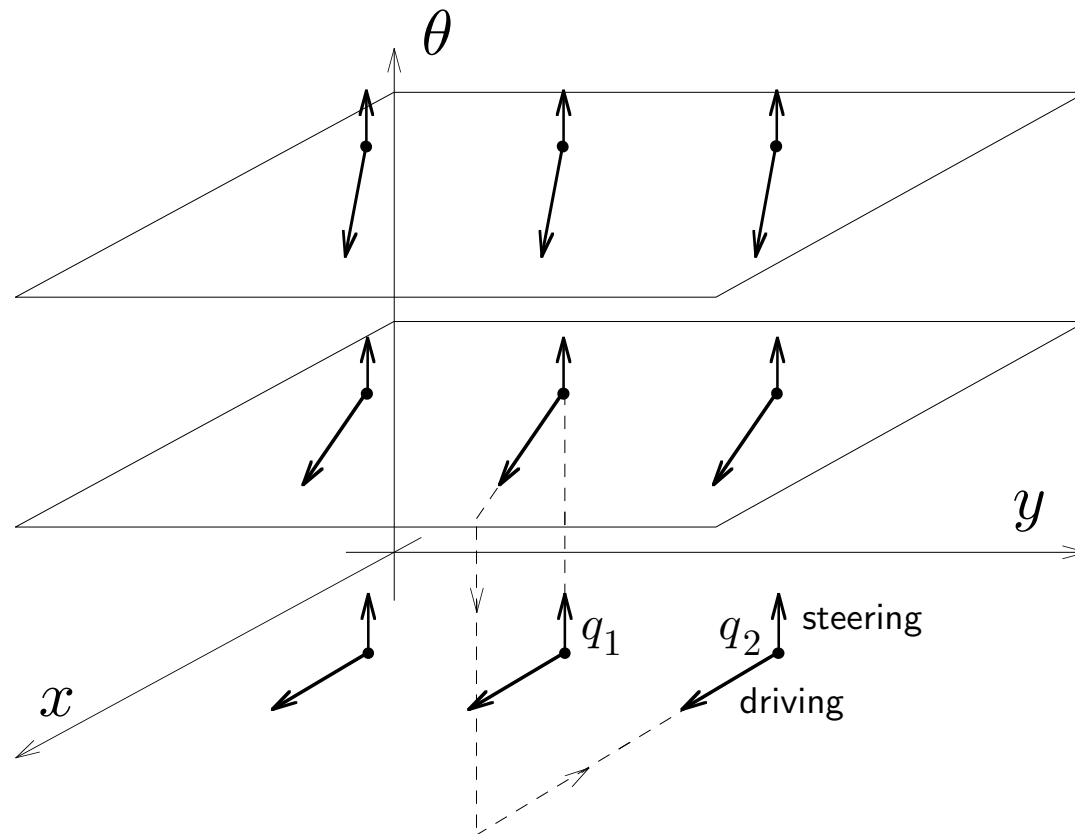


- generalized coordinates $q = (x, y, \theta)$
- **pure rolling** nonholonomic constraint $\dot{x} \sin \theta - \dot{y} \cos \theta = 0 \quad \left(\frac{\dot{y}}{\dot{x}} = \tan \theta \right)$
- feasible velocities are contained in the null space of the constraint matrix

$$a^T(q) = (\sin \theta \quad -\cos \theta \quad 0) \quad \Longrightarrow \quad \mathcal{N}(a^T(q)) = \text{span} \left\{ \begin{pmatrix} \cos \theta \\ \sin \theta \\ 0 \end{pmatrix}, \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix} \right\}$$

- **any** configuration $q_f = (x_f, y_f, \theta_f)$ can be reached:
 1. rotate the disk until it aims at (x_f, y_f)
 2. roll the disk until it reaches (x_f, y_f)
 3. rotate the disk until its orientation is θ_f

nonholonomy **in the configuration space** of the rolling disk



- at each q , only two instantaneous directions of motion are possible
- to move from q_1 to q_2 (**parallel parking**) an appropriate **maneuver** (sequence of moves) is needed; one possibility is to follow the dashed line

a less canonical example of nonholonomy: the fifteen puzzle

1	2	3	4
5	6	7	8
9	10	11	12
13	14	15	

- generalized coordinates $q = (q_1, \dots, q_{15})$
- each q_i may assume 16 different values corresponding to the cells in the grid; **legal** configurations are obtained when $q_i \neq q_j$ for $i \neq j$
- depending on the current configuration, a limited number (2 to 4) moves are possible
- **any** configuration with an **even** number of inversions can be reached by an appropriate sequence of moves

Integrability of Kinematic Constraints

- when is a single kinematic Pfaffian constraint

$$a^T(q)\dot{q} = \sum_{j=1}^n a_j(q)\dot{q}_j = 0$$

integrable as $h(q) = 0$?

since $\dot{h}(q) = \sum_{j=1}^n \frac{\partial h}{\partial q_j} \dot{q}_j = 0$, integrability requires

$$\gamma(q)a_j(q) = \frac{\partial h}{\partial q_j}(q) \quad j = 1, \dots, n$$

with $\gamma(q) \neq 0$ **integrating factor**, or equivalently

$$\frac{\partial(\gamma a_k)}{\partial q_j} = \frac{\partial(\gamma a_j)}{\partial q_k}, \quad j, k = 1, \dots, n$$

where a system of PDE's must be solved

- for k kinematic Pfaffian constraints, one must check integrability **not only** of each constraint but **also** of independent combinations

$$\sum_{i=1}^k \gamma_i(q)a_i^T(q)\dot{q} = 0$$

even if each constraint is not integrable by itself, a subset (or even the whole set) of them may be integrable!

- if there exist $p \leq k$ functions h_i such that, $\forall q$

$$\text{span} \left\{ \frac{\partial h_1}{\partial q}(q), \dots, \frac{\partial h_p}{\partial q}(q) \right\} \subset \text{span} \{a_1^T(q), \dots, a_k^T(q)\}$$

then the system motion is restricted to the $(n-p)$ -dimensional manifold of level surfaces of the h_i 's

$$\{q : h_1(q) = c_1, \dots, h_p(q) = c_p\}$$

- motion reduction due to kinematic constraints

$$p = k \iff \text{holonomic}$$

$$0 < p < k \iff \text{partially holonomic}$$

$$p = 0 \iff \text{(completely) nonholonomic}$$

- assessing integrability is **not obvious**: complete (N&S conditions) and constructive answers are obtained by differential geometric tools

A Control Viewpoint

- holonomy/nonholonomy of constraints may be conveniently studied through a dual approach: look at the

directions in which motion is **allowed**
rather than
directions in which motion is **prohibited**

- there is a strict relationship between
capability of accessing every configuration
and
nonholonomy of the velocity constraints

- the interesting question is:

given two arbitrary points q_i and q_f ,
when does a connecting trajectory $q(t)$ exist
which satisfies the kinematic constraints?



... this is indeed a **controllability** problem!

- associate to the set of kinematic constraints a basis for their null space, i.e. a set of vectors g_j such that

$$a_i^T(q)g_j(q) = 0 \quad i = 1, \dots, k \quad j = 1, \dots, n - k$$

or in matrix form

$$A^T(q)G(q) = 0$$

- feasible trajectories of the mechanical system are the solutions $q(t)$ of

$$\dot{q} = \sum_{j=1}^m g_j(q)u_j = G(q)u \quad (*)$$

for some input $u(t) \in \mathbb{R}^m$, $m = n - k$ (u : also called **pseudovelocities**)

- (*) is a **driftless** (i.e., $u=0 \Rightarrow \dot{q}=0$) nonlinear system known as the **kinematic model** of the constrained mechanical system
- **controllability** of its whole configuration space is equivalent to **nonholonomy** of the original kinematic constraints

Dynamics versus Kinematics

- use Lagrange formalism to obtain the dynamics of a mechanical system with n degrees of freedom, subject to k Pfaffian kinematic constraints

$$A^T(q)\dot{q} = 0$$

- Lagrangian = Kinetic Energy – Potential Energy

$$\mathcal{L}(q, \dot{q}) = T(q, \dot{q}) - U(q) = \frac{1}{2} \dot{q}^T B(q) \dot{q} - U(q)$$

with inertia matrix $B(q) > 0$

- **Euler-Lagrange** equations

$$\frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{q}} \right)^T - \left(\frac{\partial \mathcal{L}}{\partial q} \right)^T = A(q)\lambda + S(q)\tau$$

where

- $S(q)$ is a $n \times m$ matrix mapping the m external inputs τ into forces/torques performing work on the generalized coordinates q ($m = n - k$)
- $\lambda \in \mathbb{R}^m$ is the vector of **Lagrange multipliers**

- the **dynamic model** of the mechanism subject to constraints is

$$\begin{aligned} B(q)\ddot{q} + n(q, \dot{q}) &= A(q)\lambda + S(q)\tau & (\diamond) \\ A^T(q)\dot{q} &= 0 \end{aligned}$$

with

$$n(q, \dot{q}) = \dot{B}(q)\dot{q} - \frac{1}{2} \left(\frac{\partial}{\partial q} (\dot{q}^T B(q) \dot{q}) \right)^T + \left(\frac{\partial U(q)}{\partial q} \right)^T$$

- to eliminate the Lagrange multipliers, being

$$A^T(q)G(q) = 0$$

multiply (\diamond) by $G^T(q)$ to obtain a reduced set of $m = n - k$ differential equations

$$G^T(q) (B(q)\ddot{q} + n(q, \dot{q})) = G^T(q)S(q)\tau$$

- assume now an hypothesis of ‘enough control’

$$\det G^T(q)S(q) \neq 0$$

- merge the kinematic and dynamic models into the **reduced state-space model**

$$\begin{aligned}\dot{q} &= G(q)v \\ M(q)\dot{v} + m(q, v) &= G^T(q)S(q)\tau\end{aligned}$$

where $v \in \mathbb{R}^m$ are the pseudovelocities and

$$\begin{aligned}M(q) &= G^T(q)B(q)G(q) > 0 \\ m(q, v) &= G^T(q)B(q)\dot{G}(q)v + G^T(q)n(q, G(q)v)\end{aligned}$$

where

$$\dot{G}(q)v = \sum_{i=1}^m \left(v_i \frac{\partial g_i}{\partial q}(q) \right) G(q)v$$

- define external input τ as a **nonlinear feedback law** from the state (q, v)

$$\tau = (G^T(q)S(q))^{-1} (M(q)a + m(q, v)) \quad (\Delta)$$

where $a \in \mathbb{R}^m$ is a vector of **pseudoaccelerations**

- in the absence of constraints, (Δ) reduces to the **computed torque** law \Rightarrow linear & decoupled closed-loop dynamics (double integrators)

- due to the presence of constraints, the resulting system is

$$\begin{aligned}\dot{q} &= G(q)v && \text{kinematic model} \\ \dot{v} &= a && \text{dynamic extension}\end{aligned}$$

- letting $x = (q, v)$ and $a = u$, the **state-space model** of the closed-loop system is rewritten in compact form as

$$\dot{x} = f(x) + g(x)u = \begin{pmatrix} G(q)v \\ 0 \end{pmatrix} + \begin{pmatrix} 0 \\ I_m \end{pmatrix} u$$

i.e., a nonlinear control system **with drift** also known as the **second-order kinematic model** of the constrained mechanism



- ★ an invertible *feedback control law* can eliminate dynamic parameters
- ★ moving from kinematics to dynamics essentially requires some *input smoothness* assumptions (need $a = \dot{v}$)
- ★ most nonholonomic problems can be addressed at a *first-order* kinematic level

More General Nonholonomic Constraints

- one may also find Pfaffian constraints of the form

$$a_i^T(q)\dot{q} = c_i, \quad i = 1, \dots, k \quad \text{or} \quad A^T(q)\dot{q} = c$$

with constant c_i

- these constraints are **differential** but **not** of a kinematic nature; for example, this form arises from conservation of an initial **non-zero** angular momentum in space robots
- the mechanism subject to constraint is transformed into an equivalent control system by describing the feasible trajectories $q(t)$ as solutions of

$$\dot{q} = f(q) + \sum_{i=1}^m g_i(q)u_i$$

i.e., a nonlinear control system **with drift**, where $g_1(q), \dots, g_m(q)$ are a basis of the null space of $A^T(q)$ and the drift vector f is computed through pseudoinversion

$$f(q) = A^\#(q)c = A(q) (A^T(q)A(q))^{-1} c$$

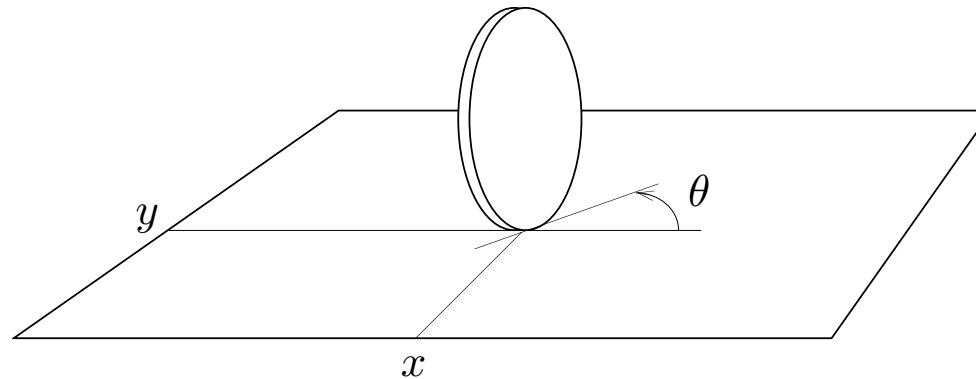
MODELING EXAMPLES

source of nonholonomic constraints on motion:

- bodies in **rolling contact without slipping**
 - wheeled mobile robots (WMRs) or automobiles (wheels rolling on the ground with no skid or slippage)
 - dextrous manipulation with multifingered robot hands (rounded fingertips on grasped objects)
- **angular momentum conservation** in multibody systems
 - robotic manipulators floating in space (with no external actuation)
 - dynamically balancing hopping robots, divers or astronauts (in flying or mid-air phases)
 - satellites with reaction (or momentum) wheels for attitude stabilization
- special **control operation**
$$\dot{q} = G(q)u \quad q \in \mathbb{R}^n \quad u \in \mathbb{R}^m \quad (m < n)$$
 - non-cyclic inversion schemes for redundant robots (m task commands for n joints)
 - floating underwater robotic systems
($m = 4$ velocity inputs for $n = 6$ generalized coords)

Wheeled Mobile Robots

unicycle



- generalized coordinates $q = (x, y, \theta)$
- nonholonomic constraint $\dot{x} \sin \theta - \dot{y} \cos \theta = 0$
- a matrix whose columns span the null space of the constraint matrix is

$$G(q) = \begin{pmatrix} \cos \theta & 0 \\ \sin \theta & 0 \\ 0 & 1 \end{pmatrix} = (g_1 \quad g_2)$$

- hence the kinematic model

$$\dot{q} = G(q)u = g_1(q)u_1 + g_2(q)u_2$$

with $u_1 = \text{driving}$, $u_2 = \text{steering}$ velocity inputs

unicycle dynamics

- define

m = mass of the unicycle

I = inertia around vertical axis at contact point

u_1 = driving force

u_2 = steering torque

- the general dynamic model

$$B(q)\ddot{q} + n(q, \dot{q}) = a(q)\lambda + S(q)\tau$$

being $B(q) = B$, $n = 0$ particularizes in this case to

$$\begin{pmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & I \end{pmatrix} \begin{pmatrix} \ddot{x} \\ \ddot{y} \\ \ddot{\theta} \end{pmatrix} = \begin{pmatrix} \sin \theta \\ -\cos \theta \\ 0 \end{pmatrix} \lambda + \begin{pmatrix} \cos \theta & 0 \\ \sin \theta & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \tau_1 \\ \tau_2 \end{pmatrix}$$

subject to $a^T(q)\dot{q} = 0$

from the reduction procedure, being

$$\begin{aligned}G(q) &= S(q) \\G^T(q)S(q) &= I_{2 \times 2} \\G^T(q)B\dot{G}(q) &= 0\end{aligned}$$

we obtain the reduced state-space model

$$\begin{aligned}\dot{q} &= G(q)v \\G^T(q)BG(q)\dot{v} &= \tau\end{aligned}$$

or the five dynamic equations

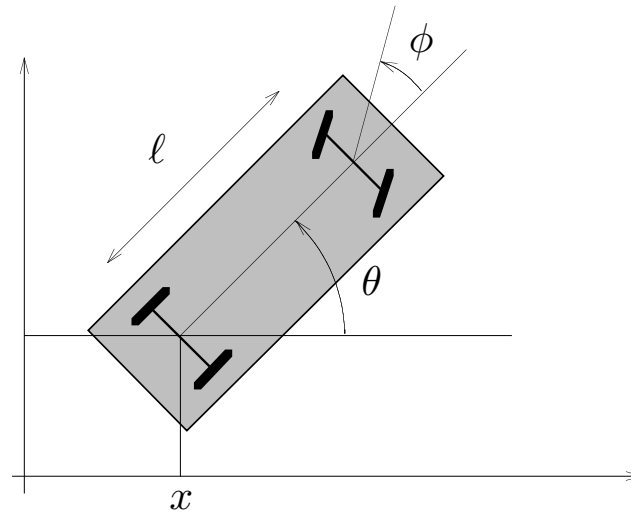
$$\begin{aligned}\dot{x} &= \cos \theta v_1 \\ \dot{y} &= \sin \theta v_1 \\ \dot{\theta} &= v_2 \\ m \dot{v}_1 &= \tau_1 \\ I \dot{v}_2 &= \tau_2\end{aligned}$$

that can be put in the form

$$\dot{X} = f(X) + g_1(X)\tau_1 + g_2(X)\tau_2$$

with $X = (x, y, \theta, v_1, v_2)$

car-like robot



- 'bicycle' model: front and rear wheels collapse into two wheels at the axle midpoints
- generalized coordinates $q = (x, y, \theta, \phi)$ ϕ : steering angle
- nonholonomic constraints

$$\begin{aligned} \dot{x}_f \sin(\theta + \phi) - \dot{y}_f \cos(\theta + \phi) &= 0 && \text{(front wheel)} \\ \dot{x} \sin \theta - \dot{y} \cos \theta &= 0 && \text{(rear wheel)} \end{aligned}$$

- being the front wheel position

$$x_f = x + l \cos \theta \quad y_f = y + l \sin \theta$$

the first constraint becomes

$$\dot{x} \sin(\theta + \phi) - \dot{y} \cos(\theta + \phi) - \dot{\theta} l \cos \phi = 0$$

the constraint matrix is

$$A^T(q) = \begin{pmatrix} \sin(\theta + \phi) & -\cos(\theta + \phi) & -\ell \cos \phi & 0 \\ \sin \theta & -\cos \theta & 0 & 0 \end{pmatrix}$$

there are two physical alternatives for the controls:

(*RD*) choosing

$$G(q) = \begin{pmatrix} \cos \theta & 0 \\ \sin \theta & 0 \\ \frac{1}{\ell} \tan \phi & 0 \\ 0 & 1 \end{pmatrix} \implies \dot{q} = g_1(q)u_1 + g_2(q)u_2$$

where $u_1 =$ **rear driving**, $u_2 =$ **steering** inputs

◇ a ‘control singularity’ at $\phi = \pm \pi/2$, where vector field g_1 diverges

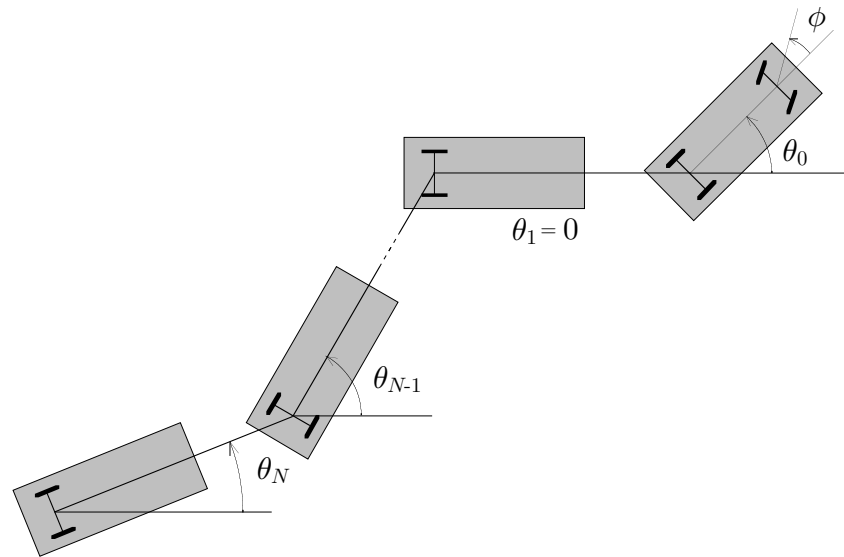
(*FD*) choosing

$$G(q) = \begin{pmatrix} \cos \theta \cos \phi & 0 \\ \sin \theta \cos \phi & 0 \\ \frac{1}{\ell} \sin \phi & 0 \\ 0 & 1 \end{pmatrix} \implies \dot{q} = g_1(q)u_1 + g_2(q)u_2$$

where $u_1 =$ **front driving**, $u_2 =$ **steering** inputs

◇ no singularities in this case!

***N*-trailer system**



- an FD car-like robot with N trailers, each hinged to the axle midpoint of the previous

- generalized coordinates $q = (x, y, \phi, \theta_0, \theta_1, \dots, \theta_N) \in \mathbb{R}^{N+4}$

x, y = position of the car rear axle midpoint

ϕ = steering angle of the car (w.r.t. car body)

θ_0 = orientation angle of the car (w.r.t. x -axis)

θ_i = orientation angle of i -th trailer (w.r.t. x)

- the car is considered as the 0-th trailer

$d_0 = \ell =$ car length

$d_i =$ i -th trailer length (hinge to hinge)

nonholonomic constraints:

steering wheel

$$\dot{x}_f \sin(\theta_0 + \phi) - \dot{y}_f \cos(\theta_0 + \phi) = 0$$

with

$$x_f = x + \ell \cos \theta_0 \quad y_f = y + \ell \sin \theta_0$$

all other wheels

$$\dot{x}_i \sin \theta_i - \dot{y}_i \cos \theta_i = 0 \quad i = 0, 1, \dots, N$$

being

$$x_i = x - \sum_{j=1}^i d_j \cos \theta_j \quad y_i = y - \sum_{j=1}^i d_j \sin \theta_j$$

the constraints become

$$\begin{aligned} \dot{x} \sin(\theta_0 + \phi) - \dot{y} \cos(\theta_0 + \phi) - \dot{\theta}_0 \ell \cos \phi &= 0 \\ \dot{x} \sin \theta_i - \dot{y} \cos \theta_i + \sum_{j=1}^i \dot{\theta}_j d_j \cos(\theta_i - \theta_j) &= 0 \quad i = 0, 1, \dots, N \end{aligned}$$

- the null space of the $N + 2$ constraints is spanned by the two columns g_1, g_2 of

$$G(q) = \begin{pmatrix} \cos \theta_0 & 0 \\ \sin \theta_0 & 0 \\ 0 & 1 \\ \frac{1}{\ell} \tan \phi & 0 \\ -\frac{1}{d_1} \sin(\theta_1 - \theta_0) & 0 \\ -\frac{1}{d_2} \cos(\theta_1 - \theta_0) \sin(\theta_2 - \theta_1) & 0 \\ \vdots & \vdots \\ -\frac{1}{d_i} \left(\prod_{j=1}^{i-1} \cos(\theta_j - \theta_{j-1}) \right) \sin(\theta_i - \theta_{i-1}) & 0 \\ \vdots & \vdots \\ -\frac{1}{d_N} \left(\prod_{j=1}^{N-1} \cos(\theta_j - \theta_{j-1}) \right) \sin(\theta_N - \theta_{N-1}) & 0 \end{pmatrix}$$

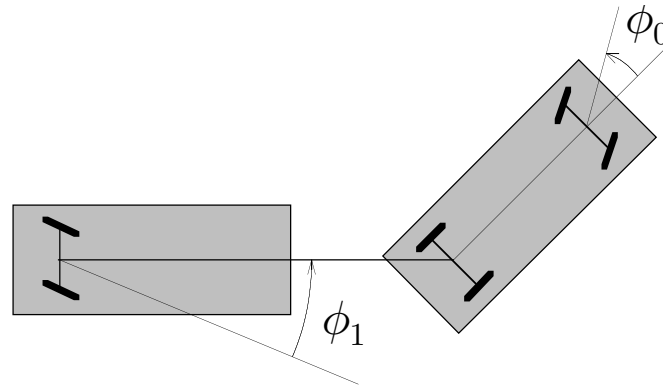
- the kinematic model is $\dot{q} = g_1(q)u_1 + g_2(q)u_2$
with $u_1 =$ **(rear) driving**, $u_2 =$ **steering** inputs for the front car
- an alternative way to derive kinematic equations

$$\begin{aligned} \dot{\theta}_i &= -\frac{1}{d_i} \sin(\theta_i - \theta_{i-1}) \nu_{i-1} \\ & \qquad \qquad \qquad i = 1, \dots, N \\ \nu_i &= \nu_{i-1} \cos(\theta_i - \theta_{i-1}) \end{aligned}$$

with $\nu_i =$ linear (forward) velocity of the i -th trailer ($\nu_0 = u_1$)

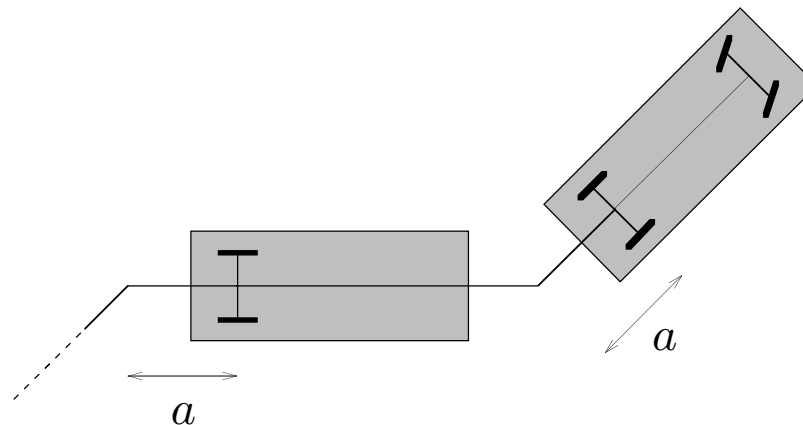
other wheeled mobile robots

- **firetruck**



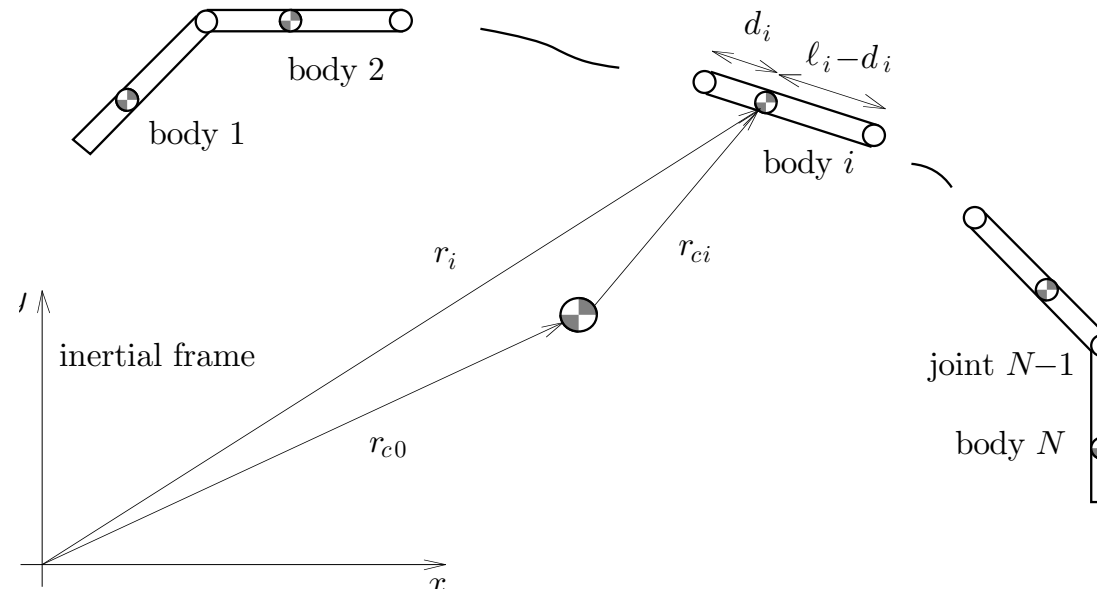
6 configuration variables, 3 differential constraints, 3 control inputs (car driving and steering, trailer steering)

- N -trailer system with **nonzero hooking**



when $a \neq 0$ and $N \geq 2$, this system **cannot** be converted in chained form (later)

Space Robots with Planar Structure



- N planar bodies actuated at the $N - 1$ joints (**internal** forces only)

- for the i -th body, let:

m_i, I_i = mass and inertia matrix

r_i, v_i = position and velocity of the center of mass

ω_i = angular velocity

- assume the center of mass of each body is located on the body axis

- no external forces (gravity), no dissipation

⇓

1. conservation of linear momentum (assumed to be initially zero)

$$\sum_{i=1}^N m_i v_i = 0 \quad \Rightarrow \quad \sum_{i=1}^N m_i r_i = m_t r_{c0}$$

i.e., two scalar **holonomic** constraints in the planar case

2. conservation of angular momentum (= zero)

$$\sum_{i=1}^N (I_i \omega_i + m_i (r_i \times v_i)) = 0$$

i.e., a scalar **nonholonomic** constraint in the planar case

- it is convenient to place the inertial frame in the center of mass of the whole system ($r_{c0} = 0$, $r_i = r_{ci}$)
- for a N -body system with kinetic energy

$$T = \frac{1}{2} \dot{q}^T B(q) \dot{q}$$

and $U = \text{constant}$, the vector of **generalized momenta** is

$$p = B(q) \dot{q} \in \mathbb{R}^N$$

- for a planar system, each component

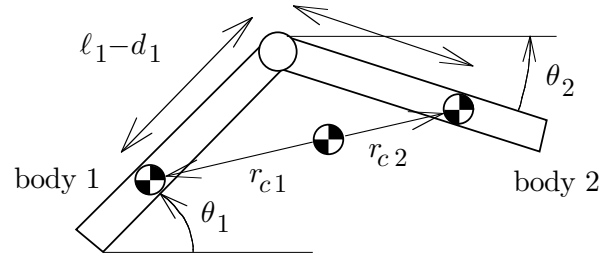
$$p_i = b_i^T(q) \dot{q}$$

represents an angular momentum along the z axis (orthogonal to the xy plane); thus, conservation of (zero) angular momentum can be expressed as a Pfaffian constraint:

$$\sum_{i=1}^N p_i = \sum_{i=1}^N b_i^T(q) \dot{q} = \mathbf{1}^T B(q) \dot{q} = A^T(q) \dot{q} = 0,$$

where $\mathbf{1} = (1, 1, \dots, 1)$

2-body space robot



from the two vector equations

$$\begin{pmatrix} r_{c1x} \\ r_{c1y} \end{pmatrix} + (\ell_1 - d_1) \begin{pmatrix} \cos \theta_1 \\ \sin \theta_1 \end{pmatrix} + d_2 \begin{pmatrix} \cos \theta_2 \\ \sin \theta_2 \end{pmatrix} = \begin{pmatrix} r_{c2x} \\ r_{c2y} \end{pmatrix}$$

$$m_1 \begin{pmatrix} r_{c1x} \\ r_{c1y} \end{pmatrix} + m_2 \begin{pmatrix} r_{c2x} \\ r_{c2y} \end{pmatrix} = 0$$

one solves for

$$\begin{pmatrix} r_{c1} \\ r_{c2} \end{pmatrix} = \begin{pmatrix} r_{c1x} \\ r_{c1y} \\ r_{c2x} \\ r_{c2y} \end{pmatrix} = \begin{pmatrix} k_{11} \begin{pmatrix} \cos \theta_1 \\ \sin \theta_1 \end{pmatrix} + k_{12} \begin{pmatrix} \cos \theta_2 \\ \sin \theta_2 \end{pmatrix} \\ k_{21} \begin{pmatrix} \cos \theta_1 \\ \sin \theta_1 \end{pmatrix} + k_{22} \begin{pmatrix} \cos \theta_2 \\ \sin \theta_2 \end{pmatrix} \end{pmatrix}$$

where (setting $m_t = m_1 + m_2$)

$$k_{11} = -\frac{m_2(\ell_1 - d_1)}{m_t} \quad k_{12} = -\frac{m_2 d_2}{m_t}$$

$$k_{21} = \frac{m_1(\ell_1 - d_1)}{m_t} \quad k_{22} = \frac{m_1 d_2}{m_t}$$

- kinetic energy of the system $T = T_1 + T_2$, with

$$T_i = \frac{1}{2} m_i \dot{r}_{ci}^T \dot{r}_{ci} + \frac{1}{2} I_{zzi} \dot{\theta}_i^2 \quad i = 1, 2$$

so that

$$T = \frac{1}{2} \begin{pmatrix} \dot{\theta}_1 & \dot{\theta}_2 \end{pmatrix} \begin{pmatrix} \bar{I}_1 & b_{12}(\theta_2 - \theta_1) \\ b_{12}(\theta_2 - \theta_1) & \bar{I}_2 \end{pmatrix} \begin{pmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{pmatrix}$$

where

$$\begin{aligned} \bar{I}_i &= I_{zzi} + m_1 k_{1i}^2 + m_2 k_{2i}^2 \quad i = 1, 2 \\ b_{12} &= (m_1 k_{11} k_{12} + m_2 k_{21} k_{22}) \cos(\theta_2 - \theta_1) \end{aligned}$$

- since T is only a function of $\phi_1 = \theta_2 - \theta_1$, the conservation of momentum can be written as the differential constraint

$$\begin{pmatrix} 1 & 1 \end{pmatrix} \begin{pmatrix} \bar{I}_1 & b_{12}(\phi_1) \\ b_{12}(\phi_1) & \bar{I}_2 \end{pmatrix} \left(\begin{pmatrix} 1 \\ 1 \end{pmatrix} \dot{\theta}_1 + \begin{pmatrix} 0 \\ 1 \end{pmatrix} \dot{\phi}_1 \right) = 0$$

from which

$$\dot{\theta}_1 = -\frac{\bar{I}_2 + b_{12}(\phi_1)}{\bar{I}_1 + \bar{I}_2 + 2b_{12}(\phi_1)} \dot{\phi}_1$$

- taking the single joint velocity $\dot{\phi}_1 = u$ as input and using as generalized coordinates

$$q = \begin{pmatrix} \theta_1 \\ \phi_1 \end{pmatrix} \quad \begin{array}{l} \text{base angle (absolute orientation)} \\ \text{relative angle (shape)} \end{array}$$

the kinematic model describing all the system feasible trajectories is

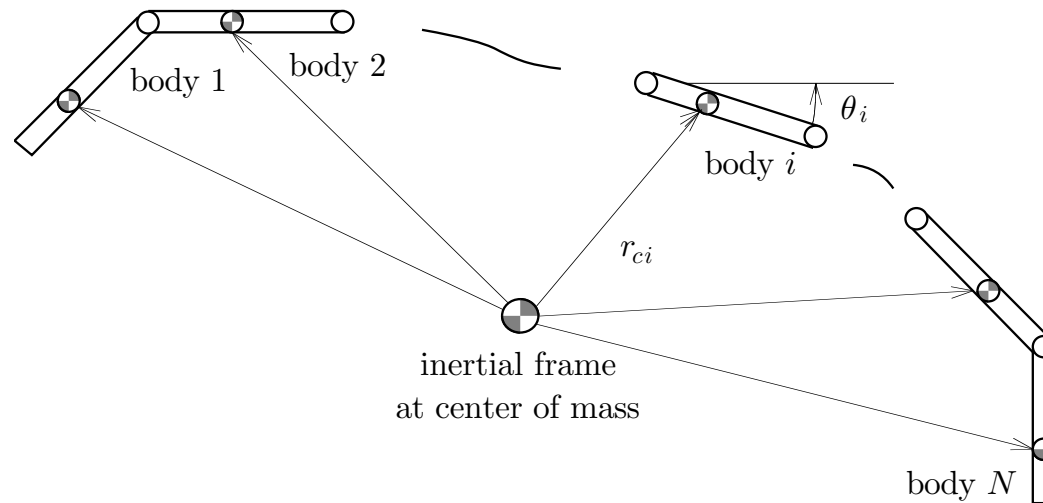
$$\dot{q} = g(q)u = \begin{pmatrix} -\frac{\bar{I}_2 + b_{12}(\phi_1)}{\bar{I}_1 + \bar{I}_2 + 2b_{12}(\phi_1)} \\ 1 \end{pmatrix} u$$

- it may be shown (see later) that such system is **not controllable**; thus, the constraint expressing conservation of the angular momentum is in this case **integrable**
in particular, if $\bar{I}_1 = \bar{I}_2$

$$\dot{\theta}_1 = -\frac{1}{2}\dot{\phi}_1 \quad \Rightarrow \quad \theta_1 = -\frac{1}{2}\phi_1 + k$$

- angular momentum conservation is a **holonomic** constraint for a planar space robot with $N = 2$ bodies
- this mechanical system **cannot be controlled** through u so as to achieve **an arbitrary pair** of absolute orientation **and** internal shape

N-body space robot



- follow the same steps as before, with the inertial reference frame placed at the system center of mass and $\theta_i =$ absolute angle of *i*-th body
- position of center of mass of *i*-th body

$$\begin{pmatrix} r_{cix} \\ r_{ciy} \end{pmatrix} = \begin{pmatrix} \sum_{j=1}^N k_{ij} \cos \theta_j \\ \sum_{j=1}^N k_{ij} \sin \theta_j \end{pmatrix}$$

where

$$k_{ij} = \begin{cases} \frac{1}{m_i} \left(\ell_j \sum_{h=1}^{j-1} m_h + (\ell_j - d_j) m_j \right) & (j < i) \\ \frac{1}{m_i} \left(d_i \sum_{h=1}^{i-1} m_h - (\ell_i - d_i) \sum_{k=i+1}^N m_k \right) & (j = i) \\ \frac{1}{m_i} \left(-\ell_j \sum_{h=j+1}^N m_h - d_j m_j \right) & (j > i) \end{cases}$$

- kinetic energy of i -th body

$$\begin{aligned}
 T_i &= \frac{1}{2} m_i \dot{r}_{ci}^T \dot{r}_{ci} + \frac{1}{2} I_{zzi} \dot{\theta}_i^2 \\
 &= \frac{1}{2} m_i \left(\sum_{h=1}^N \sum_{j=1}^N k_{ij} k_{ih} \cos(\theta_h - \theta_j) \dot{\theta}_h \dot{\theta}_j \right) + \frac{1}{2} I_{zzi} \dot{\theta}_i^2
 \end{aligned}$$

- kinetic energy of the system

$$T = \sum_{i=1}^N T_i = \frac{1}{2} \dot{\theta}^T B(\theta) \dot{\theta}$$

with elements of inertia matrix $B = \{b_{ij}(\theta_i - \theta_j)\}$

$$b_{ij} = \begin{cases} \sum_{h=1}^N m_h k_{hi} k_{hj} \cos(\theta_i - \theta_j) & i \neq j \\ \sum_{h=1}^N m_h k_{hh}^2 + I_{zzi} & i = j \end{cases}$$

depending only on **relative** angles between bodies

- let

$$\begin{aligned}
 \phi_i &= \theta_{i+1} - \theta_i & i = 1, \dots, N-1 \\
 \Rightarrow & \phi = P\theta
 \end{aligned}$$

where P is a $(N-1) \times N$ matrix

- redefine generalized coordinates as $q = (\theta_1, \phi)$

$$q = \begin{pmatrix} 1 & \mathbf{0}^T \\ & P \end{pmatrix} \theta = \begin{pmatrix} 1 & 0 & 0 & \dots & \dots \\ -1 & 1 & 0 & \dots & \dots \\ 0 & -1 & 1 & 0 & \dots \\ \dots & \dots & \dots & 0 & -1 & 1 \end{pmatrix} \theta$$

with the inverse mapping

$$\theta = \begin{pmatrix} 1 & 0 & 0 & \dots & \dots \\ 1 & 1 & 0 & \dots & \dots \\ 1 & 1 & 1 & 0 & \dots \\ \dots & \dots & \dots & \dots & \dots \\ 1 & \dots & 1 & 1 & 1 \end{pmatrix} \begin{pmatrix} \theta_1 \\ \phi \end{pmatrix} = \begin{pmatrix} 1 & S \end{pmatrix} \begin{pmatrix} \theta_1 \\ \phi \end{pmatrix}$$

where S is a $N \times (N - 1)$ matrix

- conservation of angular momentum becomes

$$\mathbf{1}^T B(\phi) (\mathbf{1}\dot{\theta}_1 + S\dot{\phi}) = 0$$

from which

$$\dot{\theta}_1 = -\frac{\mathbf{1}^T B(\phi) S}{\mathbf{1}^T B(\phi) \mathbf{1}} v$$

where $\dot{\phi} = v$ are the robot joint velocities

- the **kinematic model** of the N -body space robot is then

$$\dot{q} = \begin{pmatrix} \dot{\theta}_1 \\ \dot{\phi} \end{pmatrix} = \begin{pmatrix} s_1(\phi) & s_2(\phi) & \dots & s_{N-1}(\phi) \\ & \mathbf{I}_{N-1} & & \end{pmatrix} v$$

in which

$$s_i(\phi) = -\frac{s'_i(\phi)}{\mathbf{1}^T B(\phi) \mathbf{1}}$$

where the positive denominator is given by

$$\sum_{j=1}^N \bar{I}_j + \sum_{j=1}^N \sum_{\substack{h=1 \\ h \neq j}}^N \sum_{l=1}^N m_l k_{lj} k_{lh} \cos \left(\sum_{r=h}^{j-1} \phi_r \right)$$

with

$$\bar{I}_j = I_{zzj} + \sum_{h=1}^N m_h k_{hj}^2$$

and the numerator is

$$s'_i(\phi) = \sum_{j=i+1}^N \left(\bar{I}_j + \sum_{h=1}^N \sum_{l=1}^N m_l k_{lj} k_{lh} \cos \left(\sum_{r=h}^{j-1} \phi_r \right) \right)$$

e.g., in the case of $N = 3$ bodies

$$\begin{aligned} s'_1 &= \bar{I}_2 + \bar{I}_3 + h_{12} \cos \phi_1 + 2h_{23} \cos \phi_2 + h_{13} \cos(\phi_1 + \phi_2) \\ s'_2 &= \bar{I}_3 + h_{23} \cos \phi_2 + h_{13} \cos(\phi_1 + \phi_2) \end{aligned}$$

and

$$\mathbf{1}^T B(\phi) \mathbf{1} = \bar{I}_1 + \bar{I}_2 + \bar{I}_3 + 2(h_{12} \cos \phi_1 + h_{23} \cos \phi_2 + h_{13} \cos(\phi_1 + \phi_2))$$

with

$$\begin{aligned} \bar{I}_1 &= m_1 k_{11}^2 + m_2 k_{21}^2 + m_3 k_{31}^2 + I_{zz1} \\ \bar{I}_2 &= m_1 k_{12}^2 + m_2 k_{22}^2 + m_3 k_{32}^2 + I_{zz2} \\ \bar{I}_3 &= m_1 k_{13}^2 + m_2 k_{23}^2 + m_3 k_{33}^2 + I_{zz3} \\ h_{12} &= m_1 k_{11} k_{12} + m_2 k_{21} k_{22} + m_3 k_{31} k_{32} \\ h_{13} &= m_1 k_{11} k_{13} + m_2 k_{21} k_{23} + m_3 k_{31} k_{33} \\ h_{23} &= m_1 k_{12} k_{13} + m_2 k_{22} k_{23} + m_3 k_{32} k_{33} \end{aligned}$$

with the k_{ij} 's and m_t depending on the inertial parameters

- the dynamic model of the N -body space robot is

$$B(\theta)\ddot{\theta} + n(\theta, \dot{\theta}) = P^T \tau$$

where $\tau =$ **torques** at the $N-1$ robot joints, with

$$\mathbf{1}^T B(\theta) \dot{\theta} = 0$$

- the **reduced dynamic model** (in the 'shape space') consists of $2N - 1$ first-order differential equations

$$\begin{aligned} \dot{\theta}_1 &= -\frac{\mathbf{1}^T B(\phi) S}{\mathbf{1}^T B(\phi) \mathbf{1}} v \\ \dot{\phi} &= v \\ \dot{v} &= M^{-1}(\phi) (-m(\phi, v) + \tau) \end{aligned}$$

where

$$\begin{aligned} M(\phi) &= PB(\phi)P^T \\ m(\phi, v) &= \dot{M}(\phi)v - \frac{1}{2} \frac{\partial}{\partial \phi} (v^T M(\phi)v) \end{aligned}$$

- the right hand side of the above is **independent** of θ_1

↓

in this case, the mechanical system is referred to as a nonholonomic **Čaplygin** system

TOOLS FROM DIFFERENTIAL GEOMETRY

- a smooth **vector field** $f : \mathbb{R}^n \mapsto T_q\mathbb{R}^n$ is a smooth mapping from each point of \mathbb{R}^n to the tangent space $T_q\mathbb{R}^n$

- if f defines the rhs of a differential equation

$$\dot{q} = f(q)$$

the **flow** $\phi_t^f(q)$ of the vector field f is the mapping which associates to each q the solution evolving from q , i.e., it satisfies

$$\frac{d}{dt} \phi_t^f(q) = f(\phi_t^f(q))$$

with the **group** property $\phi_t^f \circ \phi_s^f = \phi_{t+s}^f$

in linear systems, $f(q) = Aq$, the flow is $\phi_t^f = e^{At}$

- considering two vector fields g_1 and g_2 as in

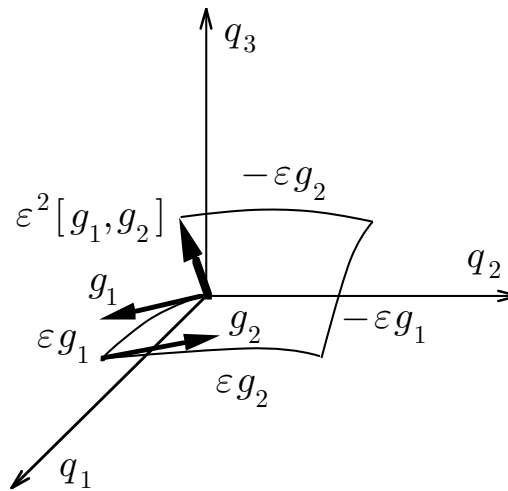
$$\dot{q} = g_1(q)u_1 + g_2(q)u_2$$

the composition of their flows (obtained by taking $u_1 = \{1, 0\}$ and $u_2 = \{0, 1\}$) is **non-commutative**

$$\phi_t^{g_1} \circ \phi_s^{g_2} \neq \phi_s^{g_2} \circ \phi_t^{g_1}$$

- starting at q_0 , an infinitesimal flow of time ϵ along g_1 , then g_2 , then $-g_1$, and finally $-g_2$, yields (R. Brockett: 'a computation everybody should do once in his life')

$$q(4\epsilon) = \phi_{\epsilon}^{-g_2} \circ \phi_{\epsilon}^{-g_1} \circ \phi_{\epsilon}^{g_2} \circ \phi_{\epsilon}^{g_1}(q_0) = q_0 + \epsilon^2 \left(\frac{\partial g_2}{\partial q} g_1(q_0) - \frac{\partial g_1}{\partial q} g_2(q_0) \right) + O(\epsilon^3)$$



- Lie bracket** of two vector fields g_1, g_2

$$[g_1, g_2](q) = \frac{\partial g_2}{\partial q} g_1(q) - \frac{\partial g_1}{\partial q} g_2(q)$$

- g_1 and g_2 **commute** if $[g_1, g_2] = 0$; moreover,

$$[g_1, g_2] = 0 \quad \Rightarrow \quad q(4\epsilon) = q_0 \quad (\text{zero net flow})$$

- **properties** of Lie brackets

$$[f, g] = -[g, f]$$

skew-symmetry

$$[f, [g, h]] + [h, [f, g]] + [g, [h, f]] = 0$$

Jacobi identity

and the **chain rule**

$$[\alpha f, \beta g] = \alpha\beta[f, g] + \alpha(L_f\beta)g - \beta(L_g\alpha)f$$

with $\alpha, \beta: \mathbb{R}^n \mapsto \mathbb{R}$ and the **Lie derivative** of α along g defined as

$$L_g\alpha(q) = \frac{\partial\alpha}{\partial q}g(q)$$

in linear single input systems, $f(q) = Aq$, $g(q) = b$,

$$\begin{aligned} [f, g] &= -Ab & [f, [f, g]] &= A^2b \\ [f, [f, [f, g]]] &= -A^3b & \dots & \end{aligned}$$

- a smooth **distribution** Δ associated with a set of smooth vector fields $\{g_1, \dots, g_m\}$ assigns to each point q a subspace of its tangent space defined as

$$\begin{aligned} \Delta &= \text{span}\{g_1, \dots, g_m\} \\ &\Downarrow \\ \Delta_q &= \text{span}\{g_1(q), \dots, g_m(q)\} \subset T_q\mathbb{R}^n \end{aligned}$$

- a distribution is **regular** if $\dim \Delta_q = \text{const}, \forall q$
- a distribution is **involutive** if it is closed under the Lie bracket operation

$$\Delta \text{ involutive} \iff \forall g_i, g_j \in \Delta \quad [g_i, g_j] \in \Delta$$
- the **involutive closure** $\bar{\Delta}$ of a distribution Δ is its closure under the Lie bracket operation
- the set of smooth vector fields on \mathbb{R}^n with the Lie bracket operation is a **Lie algebra**
- a Lie algebra is **nilpotent** if all Lie brackets of order $\geq k$ (finite integer) vanish
- a regular distribution Δ on \mathbb{R}^n of dimension k is **integrable** when there exist $n - k$ independent functions h_i such that, $\forall q$ and $\forall g_j \in \Delta$

$$L_{g_j} h_i = \frac{\partial h_i}{\partial q} g_j(q) = 0 \quad i = 1, \dots, n - k$$

- the hypersurfaces defined as the level sets

$$\{q : h_1(q) = c_1, \dots, h_{n-k}(q) = c_{n-k}\}$$

are **integral manifolds** of Δ

Frobenius Theorem

a regular distribution is integrable if and only if it is involutive

- \Rightarrow a distribution of dimension 1 (i.e., associated to a single vector field) is **always** integrable
- the proof of sufficiency is constructive
- if the distribution Δ of dimension k is involutive, then its integral manifolds (level sets of functions h_i) are **leaves** of a **foliation** of \mathbb{R}^n

e.g. the distribution $\Delta = \text{span}\{g_1, g_2\}$ with

$$g_1(q) = \begin{pmatrix} 1 \\ q_2 \\ 0 \end{pmatrix} \quad g_2(q) = \begin{pmatrix} 1 \\ 0 \\ q_3 \end{pmatrix}$$

is involutive, because

$$[g_1, g_2](q) = 0$$

it induces a foliation of \mathbb{R}^3 according to

$$q_1 - \log(q_2 q_3) = c \quad c \in \mathbb{R}$$

Integrability of Pfaffian Constraints

- a smooth **one-form** is a mapping $a^T: \mathbb{R}^n \mapsto T_q^* \mathbb{R}^n$, the dual space of linear forms on $T_q \mathbb{R}^n$

NB: one forms are represented in local coordinates as **row vectors** (hence the transpose notation!)

$$a^T(q) = (a_1(q) \quad a_2(q) \quad \dots \quad a_n(q))$$

- an **exact one-form** ω^T is the differential of a smooth function h

$$\omega^T = \frac{\partial h}{\partial q} = \left(\frac{\partial h}{\partial q_1} \quad \frac{\partial h}{\partial q_2} \quad \dots \quad \frac{\partial h}{\partial q_n} \right)$$

- a smooth **codistribution** A^T assigns to each point q a subspace of the dual of its tangent space and can be defined by a set of smooth one-forms a_i^T

$$\begin{aligned} A^T &= \text{span} \{a_1^T, \dots, a_k^T\} \\ &\Updownarrow \\ A_q^T &= \text{span} \{a_1^T(q), \dots, a_k^T(q)\} \subset T_q^* \mathbb{R}^n \end{aligned}$$

- **distribution annihilating a codistribution**

given a set of smooth independent one-forms

$$a_i^T(q) \quad i = 1, \dots, k$$

which define a codistribution A^T , there exist smooth independent vector fields

$$g_j(q) \quad j = 1, \dots, n - k = m$$

defining a distribution $\Delta = (A^T)^\perp$ such that

$$a_i^T(q) \cdot g_j(q) = 0 \quad \forall i, j$$

i.e., distribution Δ annihilates codistribution A^T



A set of Pfaffian constraints is integrable
if and only its annihilating distribution is involutive

CONTROL PROPERTIES

Controllability of Nonholonomic Systems

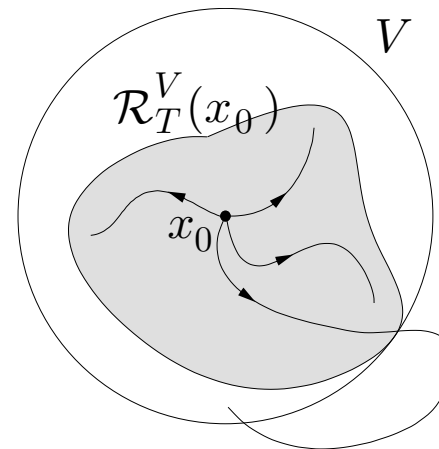
consider a nonlinear control system

$$\dot{x} = f(x) + \sum_{j=1}^m g_j(x)u_j \quad (\text{NCS})$$

with state $x \in \mathcal{M} \simeq \mathbb{R}^n$, and input in the class \mathcal{U} of piecewise-continuous time functions

- denote its unique solution at time $t \geq 0$ by $x(t, 0, x_0, u)$, with input $u(\cdot)$, and $x(0) = x_0$
- (NCS) is **controllable** if $\forall x_1, x_2 \in \mathcal{M}, \exists T < \infty, \exists u: [0, T] \rightarrow \mathcal{U} : x(T, 0, x_1, u) = x_2$
- the set of states **reachable** from x_0 **within** time $T > 0$, with trajectories contained in a neighborhood V of x_0 , is denoted by

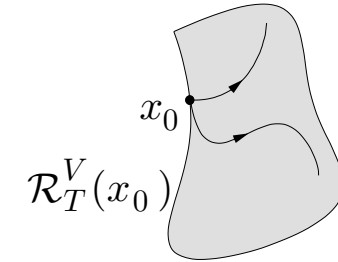
$$\mathcal{R}_T^V(x_0) = \bigcup_{\tau \leq T} \mathcal{R}^V(x_0, \tau)$$



where $\mathcal{R}^V(x_0, \tau) = \{x \in \mathcal{M} \mid x(\tau, 0, x_0, u) = x, \forall t \in [0, \tau], x(t, 0, x_0, u) \in V\}$

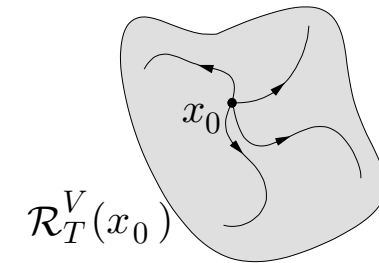
- (NCS) is **locally accessible** (LA) from x_0 if $\forall V$, a neighborhood of x_0 , and $\forall T > 0$

$$\mathcal{R}_T^V(x_0) \supset \Omega, \quad \text{with } \Omega \text{ some non-empty open set}$$



- (NCS) is **small-time locally controllable** (STLC) from x_0 if $\forall V$, a neighborhood of x_0 , and $\forall T > 0$

$$\mathcal{R}_T^V(x_0) \supset \Psi, \quad \text{with } \Psi \text{ some neighborhood of } x_0$$



- STLC \Rightarrow controllability \Rightarrow LA (not vice versa)
- LA is checked through an algebraic test
 - let $\bar{\mathcal{C}}$ be the involutive closure of the distribution associated with $\{f, g_1, g_2, \dots, g_m\}$
 - **Chow Theorem** (1939): (NCS) is LA from x_0 if and only if

$$\dim \bar{\mathcal{C}}(x_0) = n \quad \text{accessibility rank condition}$$
 - an algorithmic test:

$$\bar{\mathcal{C}} = \text{span} \left\{ v \in \bigcup_{k \geq 0} \mathcal{C}^k \right\} \quad \text{with} \quad \begin{cases} \mathcal{C}^0 = \text{span} \{f, g_1, \dots, g_m\} \\ \mathcal{C}^k = \mathcal{C}^{k-1} + \text{span} \{[f, v], [g_j, v], j = 1, \dots, m : v \in \mathcal{C}^{k-1}\} \end{cases}$$

- only **sufficient** conditions exists for STLC , e.g., [Sussmann 1987]
- however, for driftless control systems:

$$\text{LA} \iff \text{controllability} \iff \text{STLC}$$

- this equivalence holds also whenever

$$f(x) \in \text{span} \{g_1(x), \dots, g_m(x)\} \quad \forall x \in \mathcal{M}$$

(‘trivial’ drift)

- if the driftless control system

$$\dot{x} = \sum_{i=1}^m g_i(x) u_i$$

is controllable, then its **dynamic extension**

$$\begin{aligned} \dot{x} &= \sum_{i=1}^m g_i(x) v_i \\ \dot{v}_i &= u_i \quad i = 1, \dots, m \end{aligned}$$

is also controllable (and vice versa)

- in the linear case $\dot{x} = Ax + \sum_{j=1}^m b_j u_j = Ax + Bu$, all controllability definitions are equivalent and the associated tests reduce to the well-known Kalman rank condition:

$$\text{rank} (B \quad AB \quad A^2B \quad \dots \quad A^{n-1}B) = n$$

- a controllability test is a nonholonomy test!

a set of k Pfaffian constraints $A(q)\dot{q} = 0$ is nonholonomic if and only if the associated kinematic model

$$\dot{q} = G(q)u = \sum_{i=1}^m g_i(q)u_i \quad m = n - k$$

is controllable, that is

$$\dim \bar{\mathcal{C}} = n$$

being $\bar{\mathcal{C}}$ the involutive closure of the distribution associated with g_1, \dots, g_m



for a nonholonomic system, it is always possible to design **open-loop** commands that drive the system from any state to any other state (**nonholonomic path planning**)

Stabilizability of Nonholonomic Systems

for a nonlinear control system

$$\dot{x} = f(x) + \sum_{j=1}^m g_j(x)u_j = f(x) + g(x)u$$

one would like to build a **feedback control** law of the form

$$u = \alpha(x) + \beta(x)v$$

in such a way that either

- a) a desired closed-loop equilibrium point x_e is made asymptotically stable, or
 - b) a desired feasible closed-loop trajectory $x_d(t)$ is made asymptotically stable
- feedback laws are essential in motion control to counteract the presence of disturbances as well as modeling inaccuracies
 - in linear systems, controllability directly implies asymptotic (actually, exponential) stabilizability at x_e by **smooth** (actually, linear) state feedback

$$\alpha(x) = K(x - x_e)$$

- if the linear approximation of the system at x_e

$$\dot{\delta x} = A\delta x + B\delta u \quad \delta x = x - x_e, \delta u = K\delta x$$

is controllable, then the original system can be locally smoothly stabilized at x_e (a **sufficient** condition)

- in the presence of **uncontrollable eigenvalues at zero**, nothing can be concluded (except that smooth exponential stability is not achievable)
- for kinematic models of nonholonomic systems $\dot{q} = G(q)u$, the linear approximation around x_e has **always** uncontrollable eigenvalues at zero since

$$A \equiv 0 \quad \text{and} \quad \text{rank } B = \text{rank } G(q_e) = m < n$$

- however, there are **necessary** conditions for the existence of a C^0 -stabilizing state feedback law (next slide)
- whenever these conditions fail, two alternatives are left:
 - a) **discontinuous feedback** $u = \alpha(x), \alpha \in \bar{C}^0$
 - b) **time-varying feedback** $u = \alpha(x, t), \alpha \in C^1$

Brockett stabilization theorem (1983)

if the system

$$\dot{x} = f(x, u)$$

is locally asymptotically C^1 -stabilizable at x_e , then the image of the map

$$f : \mathcal{M} \times \mathcal{U} \rightarrow \mathbb{R}^n$$

contains some **neighborhood** of x_e (a **necessary** condition)

a special case: the **driftless** system

$$\dot{x} = \sum_{i=1}^m g_i(x) u_i$$

with linearly independent vectors $g_i(x_e)$, i.e.,

$$\text{rank} (g_1(x_e) \quad g_2(x_e) \quad \dots \quad g_m(x_e)) = m$$

is locally asymptotically C^1 -stabilizable at x_e **if and only if** $m \geq n$



nonholonomic mechanical systems
(either in kinematic or dynamic form)
cannot be stabilized at a point by smooth feedback

Classification of Nonholonomic Distributions

- the equivalence between a set of Pfaffian constraints

$$a_i^T(q)\dot{q} = 0 \quad i = 1, \dots, k$$

and the associated kinematic model

$$\dot{q} = \sum_{j=1}^m g_j(q)u_j \quad m = n - k$$

i.e., in matrix format

$$A^T(q)\dot{q} = 0 \quad \iff \quad \dot{q} = G(q)u$$

in the light of controllability (LA) conditions gives

$\dim \bar{\mathcal{C}} = n$	\iff	completely nonholonomic constraints (distribution)
$m < \dim \bar{\mathcal{C}} < n$	\iff	partially nonholonomic constraints (distribution)
$\dim \bar{\mathcal{C}} = m$	\iff	holonomic constraints (distribution)

- Frobenius Theorem \Rightarrow

if $\bar{\mathcal{C}}$ is regular of dimension $n - p$, there exist p functions h_j such that

$$h_j(q) = c_j \quad (j = 1, \dots, p) \iff a_i^T(q)\dot{q} = 0 \quad (i = 1, \dots, k)$$

- one may show that the **complexity** of the path planning problem is related to the level of Lie bracketing needed to span \mathbb{R}^n



classify nonholonomic systems accordingly

- the **filtration** $\{\mathcal{C}_i\}$ generated by the distribution $\mathcal{C} = \text{span}\{g_1, \dots, g_m\}$ is defined as

$$\begin{aligned} \mathcal{C}_1 &= \mathcal{C} \\ \mathcal{C}_i &= \mathcal{C}_{i-1} + [\mathcal{C}_1, \mathcal{C}_{i-1}] \quad i > 2 \end{aligned}$$

where

$$[\mathcal{C}_1, \mathcal{C}_{i-1}] = \text{span}\{[g_j, v] : g_j \in \mathcal{C}_1, v \in \mathcal{C}_{i-1}\}$$

- a filtration is **regular** in a neighborhood $V(q_0)$ if $\dim \mathcal{C}_i(q) = \dim \mathcal{C}_i(q_0), \quad \forall q \in V(q_0)$
- if $\{\mathcal{C}_i\}$ is regular, the **degree of nonholonomy** of \mathcal{C} is the smallest integer κ such that

$$\dim \mathcal{C}_{\kappa+1} = \dim \mathcal{C}_\kappa$$

- \Rightarrow nonholonomy conditions in terms of κ : a set of k Pfaffian constraints is
 1. completely nonholonomic if $\dim \mathcal{C}_\kappa = n$
 2. partially nonholonomic if $m < \dim \mathcal{C}_\kappa < n$
 3. holonomic if $\dim \mathcal{C}_\kappa = m$ ($m = n - k$)

Examples of Classification

- **unicycle kinematics** ($n = 3$)

$$g_1 = \begin{pmatrix} \cos \theta \\ \sin \theta \\ 0 \end{pmatrix} \quad g_2 = \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix} \quad g_3 = [g_1, g_2] = \begin{pmatrix} -\sin \theta \\ \cos \theta \\ 0 \end{pmatrix}$$

degree of nonholonomy $\kappa = 2$, $\dim \bar{\mathcal{C}} = 3$ for all q

- **unicycle dynamics** ($n = 5$)

$$f = \begin{pmatrix} \cos \theta v_1 \\ \sin \theta v_1 \\ v_2 \\ 0 \\ 0 \end{pmatrix} \quad g_1 = \begin{pmatrix} 0 \\ 0 \\ 0 \\ 1/m \\ 0 \end{pmatrix} \quad g_2 = \begin{pmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 1/I \end{pmatrix}$$

$$[g_1, f] = \begin{pmatrix} \cos \theta/m \\ \sin \theta/m \\ 0 \\ 0 \\ 0 \end{pmatrix} \quad [g_2, f] = \begin{pmatrix} 0 \\ 0 \\ 1/I \\ 0 \\ 0 \end{pmatrix} \quad [g_2, [f, [g_1, f]]] = \begin{pmatrix} -\sin \theta/mI \\ \cos \theta/mI \\ 0 \\ 0 \\ 0 \end{pmatrix}$$

degree of nonholonomy $\kappa = 3$; satisfies both the LA and STLC conditions since

$$g_1 \quad g_2 \quad [g_1, f] \quad [g_2, f] \quad [g_2, [f, [g_1, f]]]$$

span \mathbb{R}^5 , and the sequence is 'good' [Sussmann]

- **car-like robot (RD)** ($n = 4$)

$$g_1 = \begin{pmatrix} \cos \theta \\ \sin \theta \\ \tan \phi / \ell \\ 0 \end{pmatrix} \quad g_2 = \begin{pmatrix} 0 \\ 0 \\ 0 \\ 1 \end{pmatrix}$$

$$g_3 = [g_1, g_2] = \begin{pmatrix} 0 \\ 0 \\ -1/\ell \cos^2 \phi \\ 0 \end{pmatrix}$$

$$g_4 = [g_1, g_3] = \begin{pmatrix} -\sin \theta / \ell \cos^2 \phi \\ \cos \theta / \ell \cos^2 \phi \\ 0 \\ 0 \end{pmatrix}$$

degree of nonholonomy $\kappa = 3$, $\dim \bar{\mathcal{C}} = 4$ away from the singularity at $\phi = \pm\pi/2$ of g_1

- **car-like robot (FD)** ($n = 4$)

$$g_1 = \begin{pmatrix} \cos \theta \cos \phi \\ \sin \theta \cos \phi \\ \sin \phi / \ell \\ 0 \end{pmatrix} \quad g_2 = \begin{pmatrix} 0 \\ 0 \\ 0 \\ 1 \end{pmatrix}$$

$$g_3 = [g_1, g_2] = \begin{pmatrix} \cos \theta \sin \phi \\ \sin \theta \sin \phi \\ -\cos \phi / \ell \\ 0 \end{pmatrix}$$

$$g_4 = [g_1, g_3] = \begin{pmatrix} -\sin \theta / \ell \\ \cos \theta / \ell \\ 0 \\ 0 \end{pmatrix}$$

degree of nonholonomy $\kappa = 3$, $\dim \bar{\mathcal{C}} = 4$ for all q

- **N -trailer system** ($n = N + 4$)

for a slightly modified version of this mobile robot the degree of nonholonomy is n

- **all** the previous WMRs are STLC; **none** of these is smoothly stabilizable

- **3-body space robot** ($n = 3$)

$$g_1 = \begin{pmatrix} s_1(\phi) \\ 1 \\ 0 \end{pmatrix} \quad g_2 = \begin{pmatrix} s_2(\phi) \\ 0 \\ 1 \end{pmatrix}$$

$$g_3 = [g_1, g_2] = \begin{pmatrix} \frac{\partial s_2(\phi)}{\partial \phi_1} - \frac{\partial s_1(\phi)}{\partial \phi_2} \\ 0 \\ 0 \end{pmatrix}$$

but $g_3 = 0$ for some combinations of ϕ_1 and ϕ_2

- the filtration is not regular: thus, the degree of nonholonomy is not well defined
- using higher order brackets, $\dim \bar{\mathcal{C}} = 3$ for all q and the system is controllable

- **N -body space robot dynamics** ($n = 2N - 1$)

the system satisfies the conditions for LA, STLC, but not the necessary condition for stabilizability via C^1 -feedback

NONHOLONOMIC MOTION PLANNING

- the objective is to build a sequence of **open-loop** input commands that steer the system from q_i to q_f satisfying the nonholonomic constraints
- the degree of nonholonomy gives a good measure of the complexity of the steering algorithm
- there exist **canonical** model structures for which the steering problem can be solved efficiently
 - chained form
 - power form
 - Caplygin form
- interest in the **transformation** of the original model equation into one of these forms
- such model structures allow also a simpler design of feedback stabilizers (necessarily, non-smooth or time-varying)
- we limit the analysis to the case of systems **with two inputs**, where the three above forms are equivalent (via a coordinate transformation)

Chained Forms [Murray and Sastry 1993]

- a $(2, n)$ **chained form** is a two-input driftless control system

$$\dot{z} = g_1(z)v_1 + g_2(z)v_2$$

in the following form

$$\begin{aligned}\dot{z}_1 &= v_1 \\ \dot{z}_2 &= v_2 \\ \dot{z}_3 &= z_2 v_1 \\ \dot{z}_4 &= z_3 v_1 \\ &\vdots \\ \dot{z}_n &= z_{n-1} v_1\end{aligned}$$

- denoting repeated Lie brackets as $\text{ad}_{g_1}^k g_2$

$$\text{ad}_{g_1} g_2 = [g_1, g_2] \quad \text{ad}_{g_1}^k g_2 = [g_1, \text{ad}_{g_1}^{k-1} g_2]$$

one has

$$g_1 = \begin{pmatrix} 1 \\ 0 \\ z_2 \\ z_3 \\ \vdots \\ z_{n-1} \end{pmatrix} \quad g_2 = \begin{pmatrix} 0 \\ 1 \\ 0 \\ 0 \\ \vdots \\ 0 \end{pmatrix} \quad \Rightarrow \quad \text{ad}_{g_1}^k g_2 = \begin{pmatrix} 0 \\ \vdots \\ (-1)^k \\ \vdots \\ 0 \end{pmatrix}$$

in which $(-1)^k$ is the $(k+2)$ -th entry

- a one-chain system is **completely nonholonomic (controllable)** since the n vectors

$$\{g_1, g_2, \dots, \text{ad}_{g_1}^i g_2, \dots\} \quad i = 1, \dots, n - 2$$

are independent

- its degree of nonholonomy is $\kappa = n - 1$
- v_1 is called the **generating** input, z_1 and z_2 are called **base variables**
- if v_1 is (piecewise) constant, the system in chained form behaves like a (piecewise) linear system
- chained systems are a generalization of first- and second-order controllable systems for which sinusoidal steering from z_i to z_f minimizes the integral norm of the input
- different input commands can be used, e.g.
 - sinusoidal inputs
 - piecewise constant inputs
 - polynomial inputs

steering with sinusoidal inputs

- it is a two-phase method:

- I. steer the base variables z_1 and z_2 to their desired values z_{f1} and z_{f2} (in finite time)
- II. for each z_{k+2} , $k \geq 1$, steer z_{k+2} to its final value $z_{f,k+2}$ using

$$v_1 = \alpha \sin \omega t \quad v_2 = \beta \cos k\omega t$$

over one period $T = 2\pi/\omega$, where α , β are such that

$$\frac{\alpha^k \beta}{k!(2\omega)^k} = z_{f,k+2}(T) - z_{k+2}(0)$$

this guarantees $z_i(T) = z_i(0) = z_{fi}$ for $i < k$

in phase II, this step-by-step procedure adjusts one variable at a time by exploiting the closed-form integrability of the system equations under sinusoidal inputs

- phase II can be executed also all at once, choosing

$$\begin{aligned} v_1 &= a_0 + a_1 \sin \omega t \\ v_2 &= b_0 + b_1 \cos \omega t + \dots + b_{n-2} \cos(n-2)\omega t \end{aligned}$$

and solving numerically for the $n+1$ unknowns in terms of the desired variation of the $n-2$ states

steering with piecewise constant inputs

- an idea coming from multirate digital control, with the total travel time T divided in subintervals of length δ over which constant inputs are applied

$$\begin{aligned}v_1(\tau) &= v_{1,k} \\v_2(\tau) &= v_{2,k}\end{aligned}\quad \tau \in [(k-1)\delta, k\delta)$$

- it is convenient to keep v_1 always constant and take $n-1$ subintervals so that

$$T = (n-1)\delta \quad v_1 = \frac{z_{f1} - z_{01}}{T}$$

and the $n-1$ constant values of input v_2

$$v_{2,1}, v_{2,2}, \dots, v_{2,n-1}$$

are obtained solving a triangular linear system coming from the closed-form integration of the model equations

- if $z_{f1} = z_{01}$, an intermediate point must be added
- for small δ , a fast motion but with large inputs

steering with polynomial inputs

- idea similar to piecewise constant input, but with improved **smoothness** properties w.r.t. time (remember that kinematic models are controlled at the (pseudo)velocity level)
- the controls are chosen as

$$\begin{aligned}v_1 &= \text{sign}(z_{f1} - z_{01}) \\v_2 &= c_0 + c_1 t + \dots + c_{n-2} t^{n-2}\end{aligned}$$

with $T = z_{f1} - z_{01}$ and c_0, \dots, c_n obtained solving the linear system coming from the closed-form integration of the model equations

$$M(T) \begin{pmatrix} c_0 \\ c_1 \\ \vdots \\ c_{n-2} \end{pmatrix} + m(z_i, T) = \begin{pmatrix} z_{f2} \\ z_{f3} \\ \vdots \\ z_{fn} \end{pmatrix}$$

with $M(T)$ nonsingular for $T \neq 0$

- if $z_{f1} = z_{01}$, an intermediate point must be added
- for small T , a fast motion but with large inputs

transformation into chained form

- there exist necessary and sufficient conditions for transforming a control system

$$\dot{q} = g_1(q)u_1 + \dots + g_m(q)u_m \quad q \in \mathbb{R}^n$$

into chained form via input transformation and change of coordinates

$$v = \beta(q)u \quad z = T(q)$$

- for $m = 2$, $\mathcal{C} = \text{span}\{g_1, g_2\}$, define the filtrations

$$\begin{aligned} E_1 &= \mathcal{C} & F_1 &= \mathcal{C} \\ E_2 &= E_1 + [E_1, E_1] & F_2 &= F_1 + [F_1, F_1] \\ &\vdots & &\vdots \\ E_{i+1} &= E_i + [E_i, E_i] & F_{i+1} &= F_i + [F_i, F_1] \end{aligned}$$

- the system can be transformed in chained form if and only if

$$\dim E_i = \dim F_i = i + 1 \quad i = 1, \dots, n - 1$$

nonholonomic systems up to order $n = 4$ can be **always** be put in chained form!

- a simpler constructive sufficient condition: define the distributions

$$\Delta_0 = \text{span} \{g_1, g_2, \text{ad}_{g_1} g_2, \dots, \text{ad}_{g_1}^{n-2} g_2\}$$

$$\Delta_1 = \text{span} \{g_2, \text{ad}_{g_1} g_2, \dots, \text{ad}_{g_1}^{n-2} g_2\}$$

$$\Delta_2 = \text{span} \{g_2, \text{ad}_{g_1} g_2, \dots, \text{ad}_{g_1}^{n-3} g_2\}$$

if, for some open set, one has (i) $\dim \Delta_0 = n$ (ii) Δ_1, Δ_2 are involutive (iii) there exists a function h_1 such that

$$dh_1 \cdot \Delta_1 = 0 \quad dh_1 \cdot g_1 = 1$$

then the system can be transformed into chained form

- the change of coordinates is given by

$$\begin{aligned} z_1 &= h_1 \\ z_2 &= L_{g_1}^{n-2} h_2 \\ &\vdots \\ z_{n-1} &= L_{g_1} h_2 \\ z_n &= h_2 \end{aligned}$$

with h_2 independent from h_1 and such that

$$dh_2 \cdot \Delta_2 = 0$$

- the input transformation is given by

$$\begin{aligned} v_1 &= u_1 \\ v_2 &= (L_{g_1}^{n-1} h_2) u_1 + (L_{g_2} L_{g_1}^{n-2} h_2) u_2 \end{aligned}$$

WMRs in Chained Form

- **unicycle**

the change of coordinates

$$\begin{aligned}z_1 &= x \\z_2 &= \tan \theta \\z_3 &= y\end{aligned}$$

and input transformation

$$\begin{aligned}u_1 &= v_1 / \cos \theta \\u_2 &= v_2 \cos^2 \theta\end{aligned}$$

yield

$$\begin{aligned}\dot{z}_1 &= v_1 \\ \dot{z}_2 &= v_2 \\ \dot{z}_3 &= z_2 v_1\end{aligned}$$

other, globally defined transformations are possible

- **unicycle with N trailers**

the sufficient conditions are not satisfied but an 'ad hoc' transformation can be found (it starts using as (x, y) the position of the **last trailer** instead of the position of the trailing car)

- **car-like robot (RD)**

scaling first u_1 by $\cos \theta$

$$\begin{aligned}\dot{x} &= u_1 \\ \dot{y} &= u_1 \tan \theta \\ \dot{\theta} &= \frac{1}{l} u_1 \sec \theta \tan \phi \\ \dot{\phi} &= u_2\end{aligned}$$

then setting

$$\begin{aligned}z_1 &= x \\ z_2 &= \frac{1}{l} \sec^3 \theta \tan \phi \\ z_3 &= \tan \theta \\ z_4 &= y\end{aligned}$$

and

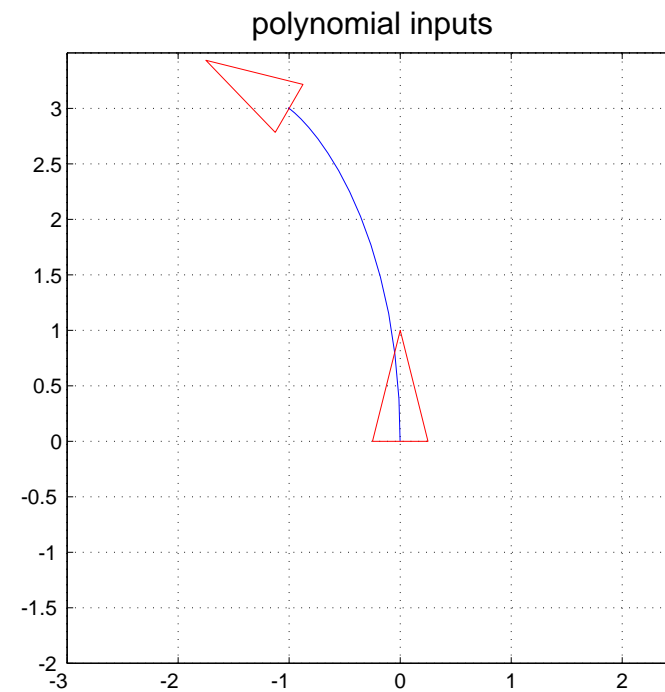
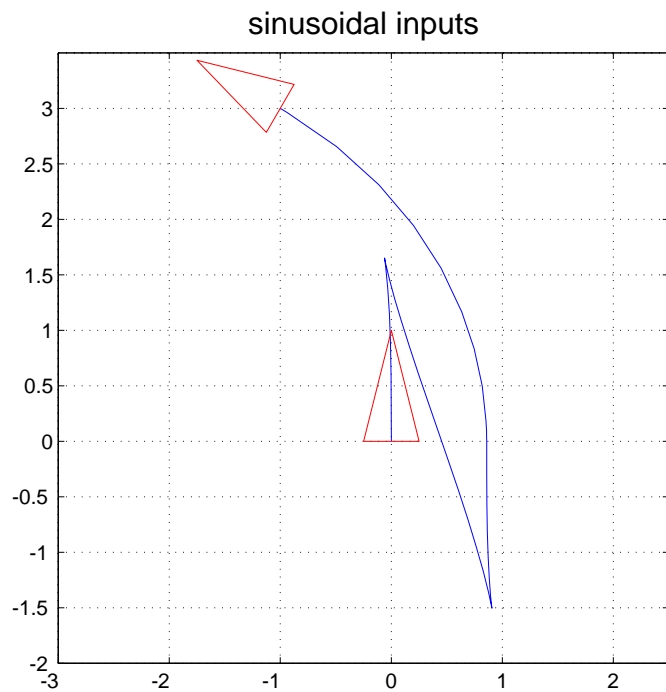
$$\begin{aligned}u_1 &= v_1 \\ u_2 &= -\frac{3}{l} v_1 \sec \theta \sin^2 \phi + \frac{1}{l} v_2 \cos^3 \theta \cos^2 \phi\end{aligned}$$

yields

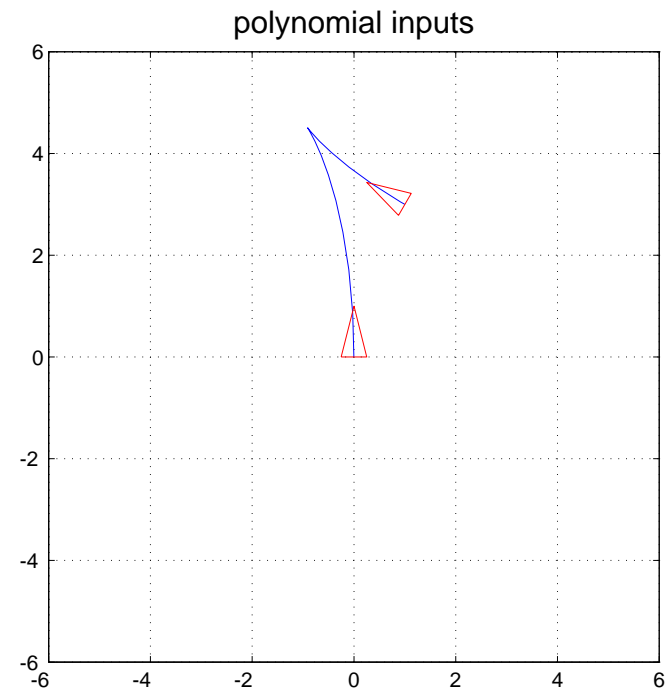
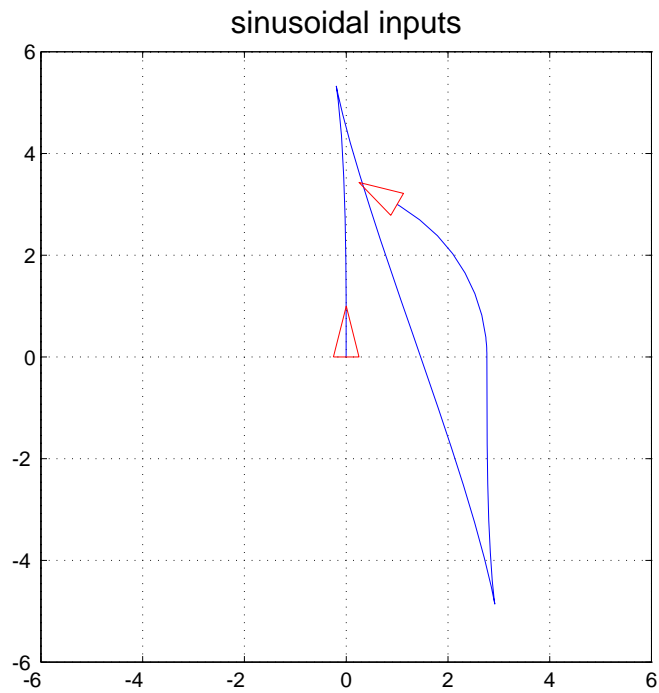
$$\begin{aligned}\dot{z}_1 &= v_1 \\ \dot{z}_2 &= v_2 \\ \dot{z}_3 &= z_2 v_1 \\ \dot{z}_4 &= z_3 v_1\end{aligned}$$

Path Planning for the Unicycle

simulation 1: $q_i = (-1, 3, 150^\circ)$, $q_f = (0, 0, 90^\circ)$



simulation 2: $q_i = (1, 3, 150^\circ)$, $q_f = (0, 0, 90^\circ)$



A General Viewpoint: Differential Flatness [Fliess *et al.* 1995]

- a nonlinear control system $\dot{z} = f(z) + G(z)v$ is **differentially flat** if there exists a set of outputs y (**flat outputs**) such that the state and the input can be expressed **algebraically** in terms of y and a certain number r of its derivatives

$$\begin{aligned}z &= z(y, \dot{y}, \ddot{y}, \dots, y^{[r]}) \\v &= v(y, \dot{y}, \ddot{y}, \dots, y^{[r]})\end{aligned}$$

- for driftless systems, flatness is equivalent to chained-form transformability; the flat outputs of a chained form are z_1, z_n (i.e., the x, y coordinates of the robot for a WMR)
- for example, for the (2,3) chained form equivalent to a unicycle, the flat outputs are z_1, z_3 ; one has

$$z_2 = \frac{\dot{z}_3}{\dot{z}_1} \quad \text{and} \quad v_1 = \dot{z}_1, \quad v_2 = \frac{\dot{z}_1 \ddot{z}_3 - \ddot{z}_1 \dot{z}_3}{\dot{z}_1^2}$$

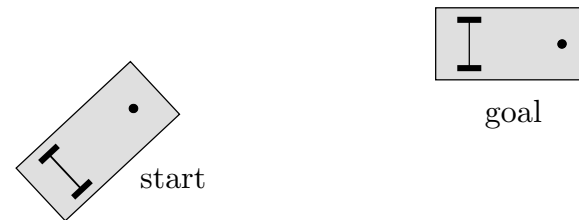
- for systems with drift, flatness is equivalent to dynamic feedback linearizability
- flatness is particularly useful for **path planning**: once the flat outputs are identified (a nontrivial task), any interpolation scheme can be used to join their initial and final values (with the appropriate boundary conditions); the evolution of the other variables as well as the control inputs are then computed through the algebraic transformations

FEEDBACK CONTROL OF NONHOLONOMIC SYSTEMS

Basic Problems

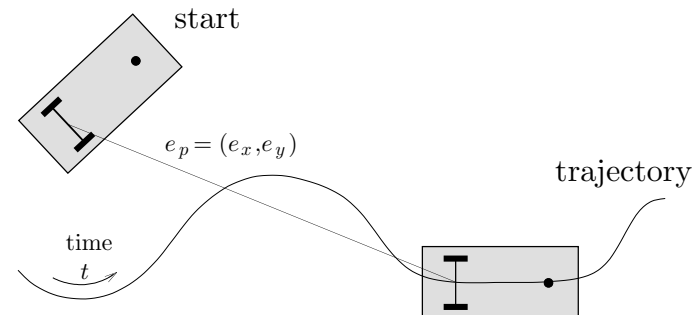
- target system: **unicycle**
 - the kinematic models of most single-body WMRs can be reduced to a unicycle
 - most of the presented design techniques can be systematically extended to chained-form transformable systems
- basic motion tasks

(a) point-to-point motion (PTPM)



(a)

(b) trajectory following (TF)



- PTPM via feedback: **posture stabilization**
 - w.l.o.g., the origin $(0, 0, 0)$ is assumed to be the desired posture
 - a **nonsquare** ($q \in \mathbb{R}^3, u \in \mathbb{R}^2$) state regulation problem
 - need to use discontinuous/time-varying feedback in view of Brockett Theorem
 - poor, erratic transient performance is often obtained (inefficient, unsafe in the presence of obstacles)
- TF via feedback: **asymptotic tracking**
 - the desired trajectory $q_d(t)$ must be feasible, i.e., comply with the nonholonomic constraints
 - a **square** ($e_p \in \mathbb{R}^2, u \in \mathbb{R}^2$) error zeroing problem
 - in this case, smooth feedback can be used because the linear approximation along a nonvanishing trajectory is controllable (see later)



asymptotic tracking is easier (and more useful) **than posture stabilization for nonholonomic systems**

Asymptotic Tracking

- a reference output trajectory $(x_d(t), y_d(t))$ is given
- control action: **feedforward** + **error feedback**
error may be defined w.r.t. the reference output (**output error**) or the associated reference state (**state error**)
- given an initial posture and a desired trajectory $(x_d(t), y_d(t))$ there is a **unique** associated state trajectory $q_d(t) = (x_d(t), y_d(t), \theta_d(t))$ which can be computed in a purely algebraic way as

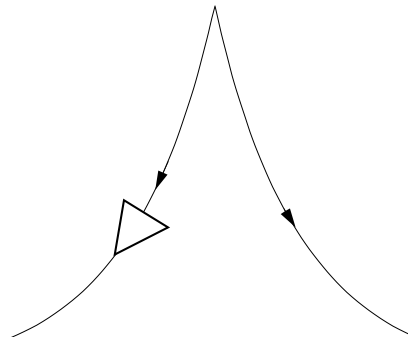
$$\theta_d(t) = \text{ATAN2}(\dot{y}_d(t), \dot{x}_d(t)) + k\pi \quad k = 0, 1$$

this is due to the fact that (x, y) is a **flat** output for the unicycle

- **feedforward command generation**: being $\theta = \text{ATAN2}(\dot{y}, \dot{x}) + k\pi$, $k = 0, 1$, we get

$$u_{d1}(t) = \pm \sqrt{\dot{x}_d^2(t) + \dot{y}_d^2(t)}$$
$$u_{d2}(t) = \frac{\ddot{y}_d(t)\dot{x}_d(t) - \ddot{x}_d(t)\dot{y}_d(t)}{\dot{x}_d^2(t) + \dot{y}_d^2(t)}$$

- the choice of sign for $u_{d1}(t)$ produces forward or backward motion
- to be exactly reproducible, $(x_d(t), y_d(t))$ should be twice differentiable
- $\theta_d(t)$ may be computed off-line and used in order to define a state error
- if $u_{d1}(\bar{t}) = 0$ for some \bar{t} (e.g., at a cusp)



neither $u_{d2}(\bar{t})$ nor $\theta_d(\bar{t})$ are defined

\Rightarrow a continuous motion is guaranteed by keeping the same orientation attained at \bar{t}^-

asymptotic tracking: controllability

linear approximation along $q_d(t) = (x_d(t), y_d(t), \theta_d(t))$

- define:

u_{d1}, u_{d2} the inputs associated to $q_d(t)$

$\tilde{q} = q - q_d$ the state tracking error

$\tilde{u}_1 = u_1 - u_{d1}$ and $\tilde{u}_2 = u_2 - u_{d2}$ the input variations

- the linear approximation along $q_d(t)$ is

$$\dot{\tilde{q}} = \begin{pmatrix} 0 & 0 & -u_{d1} \sin \theta_d \\ 0 & 0 & u_{d1} \cos \theta_d \\ 0 & 0 & 0 \end{pmatrix} \tilde{q} + \begin{pmatrix} \cos \theta_d & 0 \\ \sin \theta_d & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \tilde{u}_1 \\ \tilde{u}_2 \end{pmatrix}$$

a time-varying system

⇒ the N&S controllability condition is that the controllability Gramian is nonsingular

- a simpler analysis can be performed by ‘rotating’ the state tracking error

$$\tilde{q}_R = \begin{pmatrix} \cos \theta_d & \sin \theta_d & 0 \\ -\sin \theta_d & \cos \theta_d & 0 \\ 0 & 0 & 1 \end{pmatrix} \tilde{q}$$

according to the reference orientation θ_d

- we get

$$\dot{\tilde{q}}_R = \begin{pmatrix} 0 & u_{d2} & 0 \\ -u_{d2} & 0 & u_{d1} \\ 0 & 0 & 0 \end{pmatrix} \tilde{q}_R + \begin{pmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \tilde{u}_1 \\ \tilde{u}_2 \end{pmatrix}$$

- when the inputs u_{d1} and u_{d2} are constant, the linearization becomes time-invariant and controllable, since

$$(B \ AB \ A^2B) = \begin{pmatrix} 1 & 0 & 0 & 0 & -u_{d2}^2 & u_{d1}u_{d2} \\ 0 & 0 & -u_{d2} & u_{d1} & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \end{pmatrix}$$

has rank 3 provided that either u_{d1} or u_{d2} is nonzero

⇒ the kinematic model of the unicycle can be locally asymptotically stabilized by linear feedback along trajectories consisting of **linear or circular paths** executed at a constant velocity

(actually: the same can be proven for **any** nonvanishing trajectory)

linear control design [Samson 1992]

- designed using a (slightly different) linear approximation along the reference trajectory
- define the state tracking error e as

$$\begin{pmatrix} e_1 \\ e_2 \\ e_3 \end{pmatrix} = \begin{pmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} x_d - x \\ y_d - y \\ \theta_d - \theta \end{pmatrix}$$

- use a nonlinear transformation of velocity inputs

$$\begin{aligned} u_1 &= u_{d1} \cos e_3 - v_1 \\ u_2 &= u_{d2} - v_2 \end{aligned}$$

- the error dynamics becomes

$$\dot{e} = \begin{pmatrix} 0 & u_{d2} & 0 \\ -u_{d2} & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix} e + \begin{pmatrix} 0 \\ \sin e_3 \\ 0 \end{pmatrix} u_{d1} + \begin{pmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} v_1 \\ v_2 \end{pmatrix}$$

- linearizing around the reference trajectory, one obtains the same linear time-varying equations as before, now with state e and input (v_1, v_2)

- define the ‘linear’ feedback law

$$\begin{aligned} v_1 &= -k_1 e_1 \\ v_2 &= -k_2 \operatorname{sign}(u_{d1}(t)) e_2 - k_3 e_3 \end{aligned}$$

with gains

$$k_1 = k_3 = 2\zeta a \quad k_2 = \frac{a^2 - u_{d2}(t)^2}{|u_{d1}(t)|}$$

- the closed-loop characteristic polynomial is $(\lambda + 2\zeta a)(\lambda^2 + 2\zeta a\lambda + a^2)$, $\zeta \in (0, 1)$ $a > 0$
- a convenient **gain scheduling** is achieved letting

$$a = a(t) = \sqrt{u_{d2}^2(t) + bu_{d1}^2(t)} \quad \implies \quad k_1 = k_3 = 2\zeta \sqrt{u_{d2}^2(t) + bu_{d1}^2(t)}, \quad k_2 = b|u_{d1}(t)|$$

these gains go to zero when the state trajectory stops (and local controllability is lost)

- the actual controls are **nonlinear** and **time-varying**
- even if the eigenvalues are constant, local asymptotic stability is not guaranteed as the system is still time-varying

\Rightarrow a Lyapunov-based analysis is needed

nonlinear control design [Samson 1993]

for the previous error dynamics, define

$$\begin{aligned}v_1 &= -k_1(u_{d1}(t), u_{d2}(t)) e_1 \\v_2 &= -\bar{k}_2 u_{d1}(t) \frac{\sin e_3}{e_3} e_2 - k_3(u_{d1}(t), u_{d2}(t)) e_3\end{aligned}$$

with constant $\bar{k}_2 > 0$ and positive, continuous gain functions $k_1(\cdot, \cdot)$ and $k_3(\cdot, \cdot)$

theorem *if u_{d1} , u_{d2} , \dot{u}_{d1} \dot{u}_{d2} are bounded, and if $u_{d1}(t) \not\rightarrow 0$ or $u_{d2}(t) \not\rightarrow 0$ as $t \rightarrow \infty$, the above control globally asymptotically stabilizes the origin $e = 0$*

proof based on the Lyapunov function

$$V = \frac{\bar{k}_2}{2} (e_1^2 + e_2^2) + \frac{e_3^2}{2}$$

nonincreasing along the closed-loop solutions

$$\dot{V} = -k_1 \bar{k}_2 e_1^2 - k_3 e_3^2 \leq 0$$

$\Rightarrow \|e(t)\|$ is bounded, $\dot{V}(t)$ is uniformly continuous, and $V(t)$ tends to some limit value

\Rightarrow using Barbalat lemma, $\dot{V}(t)$ tends to zero

\Rightarrow analyzing the system equations, one can show that $(u_{d1}^2 + u_{d2}^2)e_i^2$ ($i = 1, 2, 3$) tends to zero so that, from the persistency of the trajectory, the thesis follows \blacksquare

dynamic feedback linearization [Oriolo *et al.*, 2002]

- define the output as $\eta = (x, y)$; differentiation w.r.t. time yields

$$\dot{\eta} = \begin{pmatrix} \dot{x} \\ \dot{y} \end{pmatrix} = \begin{pmatrix} \cos \theta & 0 \\ \sin \theta & 0 \end{pmatrix} \begin{pmatrix} u_1 \\ u_2 \end{pmatrix}$$

\Rightarrow cannot recover u_2 from first-order differential information

- add an integrator on the linear velocity input

$$u_1 = \xi, \quad \dot{\xi} = a \quad \Rightarrow \quad \dot{\eta} = \xi \begin{pmatrix} \cos \theta \\ \sin \theta \end{pmatrix}$$

new input a is the unicycle **linear acceleration**

- differentiating further

$$\ddot{\eta} = \begin{pmatrix} \cos \theta & -\xi \sin \theta \\ \sin \theta & \xi \cos \theta \end{pmatrix} \begin{pmatrix} a \\ u_2 \end{pmatrix}$$

- **assuming** $\xi \neq 0$, we can let

$$\begin{pmatrix} a \\ u_2 \end{pmatrix} = \begin{pmatrix} \cos \theta & -\xi \sin \theta \\ \sin \theta & \xi \cos \theta \end{pmatrix}^{-1} \begin{pmatrix} v_1 \\ v_2 \end{pmatrix}$$

obtaining

$$\ddot{\eta} = \begin{pmatrix} \ddot{\eta}_1 \\ \ddot{\eta}_2 \end{pmatrix} = \begin{pmatrix} v_1 \\ v_2 \end{pmatrix}$$

- the resulting dynamic compensator is

$$\begin{aligned}\dot{\xi} &= v_1 \cos \theta + v_2 \sin \theta \\ u_1 &= \xi \\ u_2 &= \frac{v_2 \cos \theta - v_1 \sin \theta}{\xi}\end{aligned}$$

- as the dynamic compensator is 1-dim, we have $n + 1 = 4$, equal to the total number of output differentiations

⇒ in the new coordinates

$$\begin{aligned}z_1 &= x \\ z_2 &= y \\ z_3 &= \dot{x} = \xi \cos \theta \\ z_4 &= \dot{y} = \xi \sin \theta\end{aligned}$$

the system is fully linearized and described by two chains of second-order input-output integrators

$$\begin{aligned}\ddot{z}_1 &= v_1 \\ \ddot{z}_2 &= v_2\end{aligned}$$

- the dynamic feedback linearizing controller has a potential singularity at $\xi = u_1 = 0$, i.e., when the unicycle is not rolling

a singularity in the dynamic extension process is **structural** for nonholonomic systems

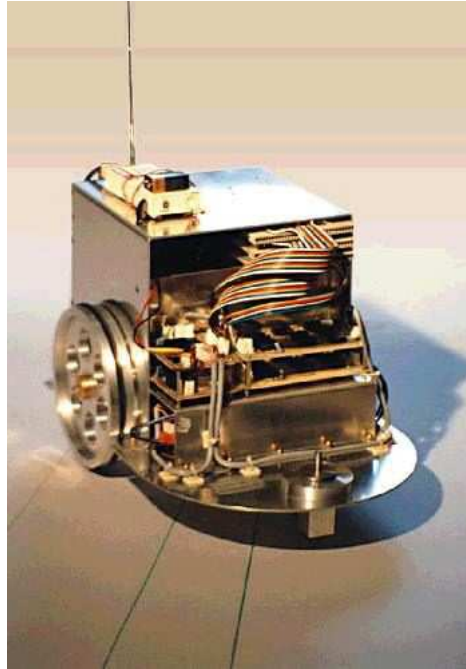
- for the (exactly) linearized system, a globally exponentially stabilizing feedback is

$$\begin{aligned} v_1 &= \ddot{x}_d(t) + k_{p1}(x_d(t) - x) + k_{d1}(\dot{x}_d(t) - \dot{x}) \\ v_2 &= \ddot{y}_d(t) + k_{p2}(y_d(t) - y) + k_{d2}(\dot{y}_d(t) - \dot{y}) \end{aligned}$$

with PD gains $k_{pi} > 0$, $k_{di} > 0$, for $i = 1, 2$

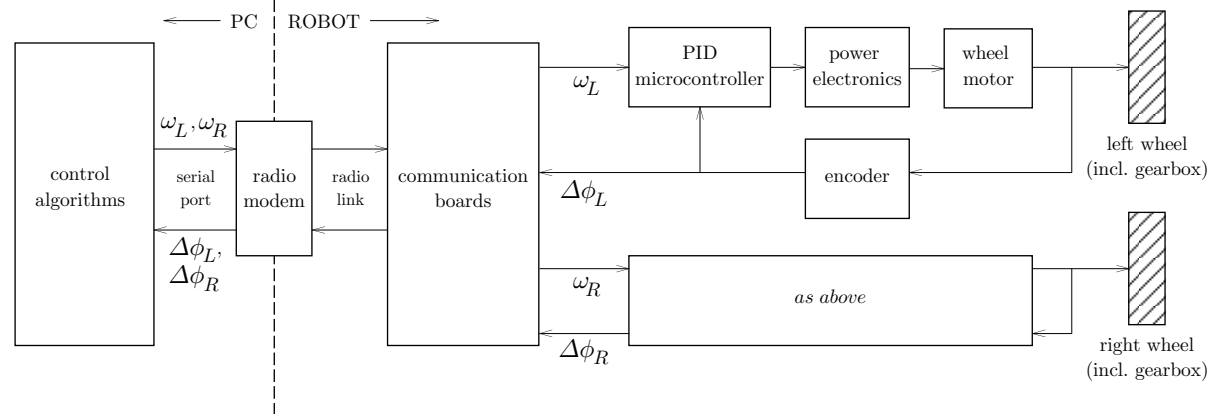
- the desired trajectory $(x_d(t), y_d(t))$ must be smooth and **persistent**, i.e., $u_{d1}^2 = \dot{x}_d^2 + \dot{y}_d^2$ must never go to zero
- cartesian transients are linear
- \dot{x} and \dot{y} can be computed as a function ξ and θ ; alternatively, one can use estimates of \dot{x} and \dot{y} obtained from odometric measurements
- for **exact tracking**, one needs $q(0) = q_d(0)$ and $\xi(0) = u_{d1}(0)$ (\Rightarrow pure feedforward)

experiments with SuperMARIO



- a two-wheel differentially-driven vehicle (with castor)
- the aluminum chassis measures $46 \times 32 \times 30.5$ cm (l/w/h) and contains two motors, transmission elements, electronics, and four 12 V batteries; total weight about 20 kg
- each wheel independently driven by a DC motor (peak torque ≈ 0.56 Nm); each motor equipped with an encoder (200 pulse/turn) and a gearbox (reduction ratio 20)
- typical nonidealities of electromechanical systems: friction, gear backlash, wheel slippage, actuator deadzone and saturation
- due to robot and motor dynamics, discontinuous velocity commands cannot be realized

two-level control architecture



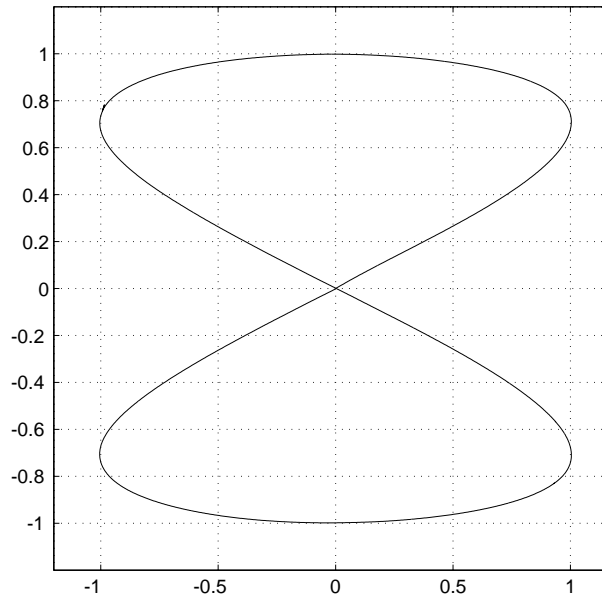
- control algorithms (with reference generation) are written in C++ and run with a sampling time of $T_s = 50$ ms on a remote server
- the PC communicates through a radio modem with the serial communication boards on the robot
- actual commands are the angular velocities ω_R and ω_L of right and left wheel (instead of driving and steering velocities u_1 and u_2):

$$u_1 = \frac{r(\omega_R + \omega_L)}{2} \quad u_2 = \frac{r(\omega_R - \omega_L)}{d}$$

with $d =$ axle length, $r =$ wheel radius

- reconstruction of the current robot state based on encoder data (**dead reckoning**)

experiments on an eight-shaped trajectory



- the reference trajectory

$$x_d(t) = \sin \frac{t}{10} \quad y_d(t) = \sin \frac{t}{20} \quad t \in [0, T]$$

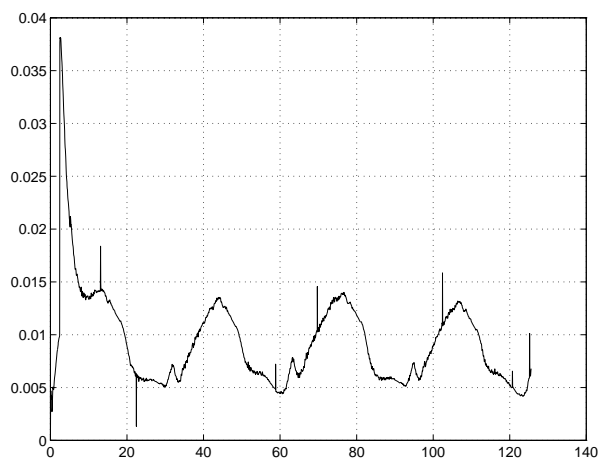
starts from the origin with $\theta_d(0) = \pi/6$ rad

- a full cycle is completed in $T = 2\pi \cdot 20 \approx 125$ s
- the reference initial velocities are

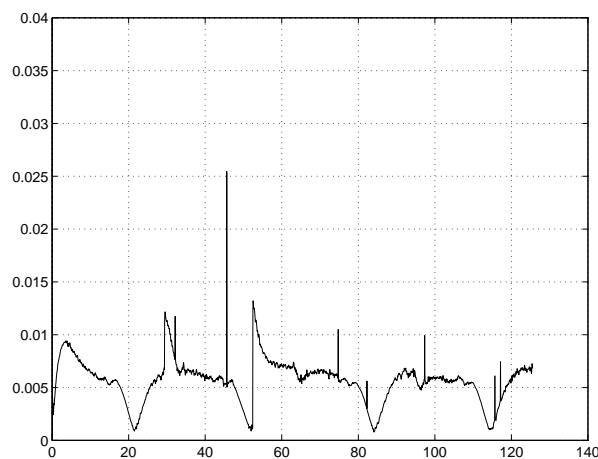
$$u_{d1}(0) \simeq 0.1118 \text{ m/s}, \quad u_{d2}(0) = 0 \text{ rad/s.}$$

experiment 1: the robot initial state is **on** the reference trajectory

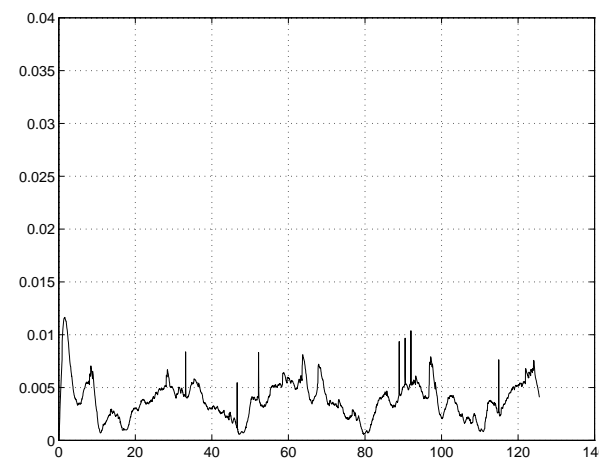
tracking error norm



linear design

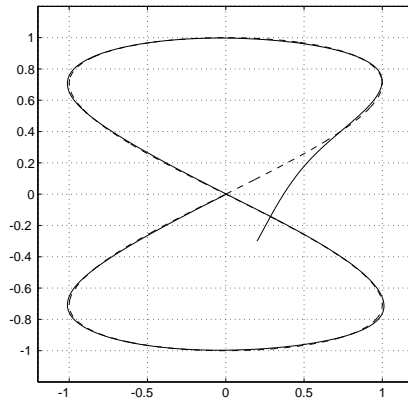


nonlinear design

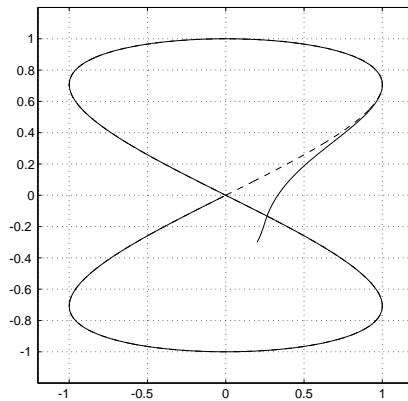
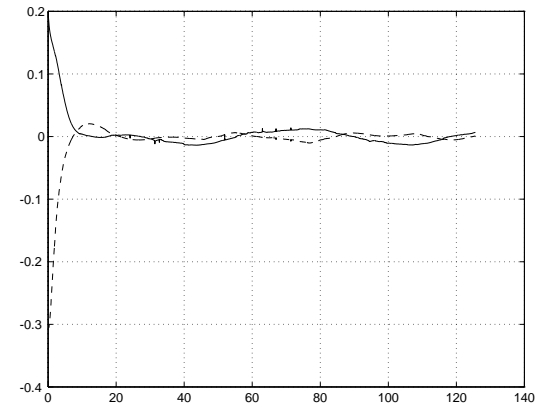


feedback linearization

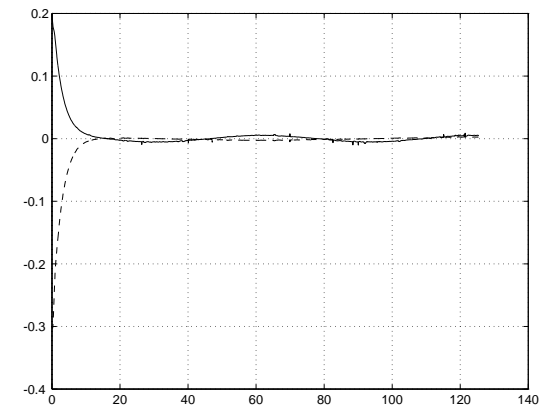
experiment 2: the robot initial state is **off** the reference trajectory



linear design



feedback linearization



Posture Stabilization: A Bird's Eye View

- the main obstruction is the non-smooth stabilizability of WMRs at a point
- two main approaches
 - **time-varying** stabilizers: an exogenous time-varying signal is injected in the controller [Samson 1991]
 - **discontinuous** stabilizers: the controller is time invariant but discontinuous at the origin [Sørdalen 1993]
- drawbacks: slow convergence (time-varying), oscillatory transient (both)
- improvements
 - **mixed time-varying/discontinuous** stabilizers [Pomet and Samson 1993; Murray and M'Closkey 1995, Lucibello and Oriolo 2001]
 - **non-Lyapunov, discontinuous** stabilizers: through coordinate transformations that circumvent Brockett's obstruction [Aicardi *et al.* 1995; Astolfi 1995] or via dynamic feedback linearization [Oriolo *et al.* 2002]
 - ↪ excellent transient performance!

OPTIMAL TRAJECTORIES FOR WMRS

(by M. Vendittelli)

- the **main objective** is to determine an optimal control law steering the kinematic model of the nonholonomic system between any two points of the configuration space
- a **first step** is to obtain a family of trajectories containing an optimal solution to the steering problem
- Pontryagin's Maximum Principle (PMP) can be used to this end providing necessary conditions for trajectories to be optimal
- characterization of optimal trajectories is not easy essentially due to the local nature of PMP
- local information needs to be completed by global study based on geometric reasoning

Minimum-Time Problems

- **objective:** compute the control law (if it exists) that steers the nonholonomic system

$$\dot{q} = G(q)u, \quad q \in \mathcal{M} \simeq \mathbb{R}^n, \quad u \in U \subset \mathbb{R}^m$$

from q_i to q_f minimizing the functional

$$J = \int_{t_i}^{t_f} dt$$

- **theorem** (existence of optimal trajectories)

under the usual assumptions for existence and uniqueness of solution of an ordinary differential equation and the additional hypothesis

$$U \text{ **compact convex** subset of } \mathbb{R}^m \quad (\Delta)$$

any two points $q_i, q_f \in \mathcal{M}$ that can be joined by an admissible trajectory can be joined by a time-optimal trajectory

- consider the **Hamiltonian**

$$H(\psi, q, u) = \langle \psi, G(q)u \rangle$$

where $\psi \in \mathbb{R}^n$ and $\langle \cdot, \cdot \rangle$ is the inner product in \mathbb{R}^n

- if $u(t) : [t_i, t_f] \rightarrow U$ is an admissible control law and $q(t) : [t_i, t_f] \rightarrow \mathcal{M}$ the corresponding trajectory,
a vector function $\psi : [t_i, t_f] \rightarrow \mathbb{R}^n$ is an **adjoint vector** for (q, u) if it satisfies

$$\dot{\psi}(t)^T = -\frac{\partial H}{\partial q}(\psi(t), q(t), u(t)) \quad \forall t \in [t_i, t_f]$$

- note that

either $\psi(t) \neq 0 \quad \forall t \in [t_i, t_f]$ (*nontrivial ψ*)

or $\psi(t) \equiv 0 \quad \forall t \in [t_i, t_f]$ (*trivial ψ*)

due to the linearity of H (\Rightarrow of $\dot{\psi}$) w.r.t. ψ

PMP for time-optimal control

consider an admissible control law $u(t)$ and the corresponding trajectory $q(t)$; a necessary condition for $q(t)$ to be time-optimal is that there exist a nontrivial adjoint vector $\psi(t)$ and a constant $\psi_0 \leq 0$ s.t.

$$H(\psi(t), q(t), u(t)) = \max_{v \in U} \{H(\psi(t), q(t), v)\} = -\psi_0 \quad (*)$$

$$\forall t \in [t_i, t_f]$$

- a control law $u(t)$ satisfying condition (*) is called an **extremal control law**
- denoting by q, ψ the trajectory and the adjoint vector corresponding to the extremal control law u , the triple (q, u, ψ) is called **extremal**
- an extremal triple (q, u, ψ) s.t. $\psi_0 = 0$ is called **abnormal**
- a control law $u(t)$ is called **singular** if there exist a nonempty subset $W \subset U$ and a nonempty interval $I \subset [t_i, t_f]$ such that

$$H(\psi(t), q(t), u(t)) = H(\psi(t), q(t), w(t))$$

$$\forall t \in I, \forall w(t) \in W$$

Application to WMRs

- target system: unicycle

$$\dot{q} = g_1(q)u_1 + g_2(q)u_2 \quad (u_1, u_2) \in U \subset \mathbb{R}^2$$

$$\text{with } q = (x, y, \theta) \quad g_1(q) = \begin{pmatrix} \cos \theta \\ \sin \theta \\ 0 \end{pmatrix} \quad g_2(q) = \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix}$$

- new terminology based on **control domains**

$$U = [-k_1, k_1] \times [-k_2, k_2] \quad \textit{unicycle}$$

$$U = \{-k_1, k_1\} \times [-k_2, k_2] \quad \textit{Reeds and Shepp's car}$$

$$U = k_1 \times [-k_2, k_2] \quad \textit{Dubins' car}$$

$$\text{with } k_1, k_2 > 0$$

unicycle and Reeds and Shepp's car are STLC

Dubins' car is controllable but not STLC

- unicycle and Dubins' car verify the conditions for existence of optimal trajectories
- Reeds and Shepp's car does not verify condition (Δ); existence of optimal trajectories has been established as a byproduct of the analysis of the optimal control problem for the unicycle

unicycle

- $(u_1, u_2) \in U = [-1, 1] \times [-1, 1]$ (w.l.o.g.)
- the corresponding Hamiltonian is

$$H = \psi_1 \cos \theta u_1 + \psi_2 \sin \theta u_1 + \psi_3 u_2$$

- it is convenient to define the **switching functions**

$$\phi_1 = \langle \psi, g_1 \rangle = \psi_1 \cos \theta + \psi_2 \sin \theta, \quad \phi_2 = \langle \psi, g_2 \rangle = \psi_3$$

and write the Hamiltonian as

$$H = \phi_1 u_1 + \phi_2 u_2$$

- the switching functions determine the sign changes of u_1, u_2 (see later)
- applying PMP

$$-\psi_0 = H(\psi(t), q(t), u(t)) = \max_{v \in U} (H(\psi(t), q(t), v)) = \max_{(v_1, v_2) \in U} (\phi_1 v_1 + \phi_2 v_2) \quad (1)$$

where

$$\dot{\psi}(t) = -\frac{\partial H}{\partial q}(\psi(t), q(t), u(t)) = -\frac{\partial}{\partial q}(\phi_1 u_1 + \phi_2 u_2)$$

extracting information from PMP

- maximization of the Hamiltonian (i.e. cond. (1)) implies that on extremal trajectories

$$u_1 = \text{sign}(\phi_1) \quad u_2 = \text{sign}(\phi_2) \quad (2)$$

where

$$\text{sign}(s) = \begin{cases} 1 & \text{if } s > 0 \\ -1 & \text{if } s < 0 \\ \text{any number in } [-1, 1] & \text{if } s = 0 \end{cases}$$

- on any subinterval of $[t_i, t_f]$ where $\phi_j \neq 0$ ($j=\{1,2\}$) u_j is **bang** (i.e. maximal or minimal)
- a necessary condition for t to be a switching time for $u_j(t)$ is that $\phi_j(t) = 0$
- if $\phi_j(t) = 0$ on a nonempty interval $I \subset [t_i, t_f]$ the corresponding control $u_j(t)$ is singular on I

- to characterize the structure of extremals

define

$$\phi_3 = \langle \psi, [g_2, g_1] \rangle$$

compute

$$\dot{\phi}_1 = u_2 \cdot \langle \psi, [g_2, g_1] \rangle = u_2 \phi_3$$

$$\dot{\phi}_2 = -u_1 \cdot \langle \psi, [g_2, g_1] \rangle = -u_1 \phi_3 \quad (3)$$

$$\dot{\phi}_3 = -u_2 \phi_1$$

- from (1), (2)

$$|\phi_1| + |\phi_2| + \psi_0 = 0 \quad (4)$$

- from $\psi \neq 0$ + controllability

$$|\phi_1| + |\phi_2| + |\phi_3| \neq 0 \quad (5)$$

- (2), (3), (4), (5) are called **Switching Structure Equations**

lemma 1 nontrivial abnormal extremals do not exist

proof: use (4), (5), (3)

lemma 2 for a nontrivial optimal extremal, ϕ_1 and ϕ_2 cannot have a common zero

proof: use (4)

lemma 3 along an extremal, $\kappa = \phi_1^2 + \phi_3^2$ is constant and $\kappa = 0 \iff \phi_1 \equiv 0$

proof: use lemma 2, (3)

lemma 4 along an extremal, either all the zeros of ϕ_1 are isolated and s.t. $\dot{\phi}_1$ exists and is $\neq 0$ or $\phi_1 \equiv 0$

proof: use lemma 3, lemma 2, (3)

↓

there exist two kinds of extremal trajectories

A trajectories with a finite number of switchings

B trajectories along which $\phi_1 \equiv 0$ and either $u_2 \equiv 1$ or $u_2 \equiv -1$

to simplify the geometric description of the extremals it is useful to introduce the following

notation

C_a arc of circle of length a

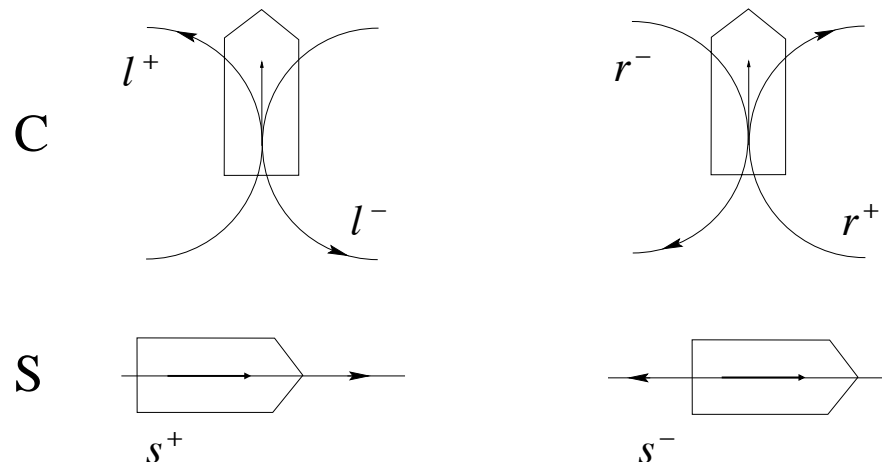
S_a straight line segment of length a

$C|C$ arcs of circle connected by a cusp

$l_a^{+(-)}$ forward (backward) left motion along the arc of length a

$r_a^{+(-)}$ forward (backward) right motion along the arc of length a

$s_a^{+(-)}$ forward (backward) motion along the straight line segment of length a



type A trajectories

the integration of the adjoint system

$$\begin{cases} \dot{\psi}_1 = -\frac{\partial H}{\partial x} = 0 \\ \dot{\psi}_2 = -\frac{\partial H}{\partial y} = 0 \\ \dot{\psi}_3 = -\frac{\partial H}{\partial \theta} = \psi_1 \sin \theta u_1 - \psi_2 \cos \theta u_1 = \psi_1 \dot{y} - \psi_2 \dot{x} \end{cases}$$

implies (w.l.o.g. $x(t_i) = y(t_i) = 0$)

- ψ_1 and ψ_2 constant
- $\psi_3(t) = \psi_3(t_i) + \psi_1 y - \psi_2 x = \phi_2(t)$
- if $\phi_1 = 0$ (switch of u_1), (1) implies $\phi_2 u_2 + \psi_0 = \psi_3 u_2 + \psi_0 = 0$

- if $u_2 = 1$ the **cusp** point is on the line

$$\mathcal{D}^+ : \psi_1 y(t) - \psi_2 x(t) + \psi_3(t_i) + \psi_0 = 0$$

- if $u_2 = -1$ the cusp point is on the line

$$\mathcal{D}^- : \psi_1 y(t) - \psi_2 x(t) + \psi_3(t_i) - \psi_0 = 0$$

- if $\phi_2 = 0$ (switch of u_2) the **inflection** point lies on the line

$$\mathcal{D}_0 : \psi_1 y(t) - \psi_2 x(t) + \psi_3(t_i) = 0$$

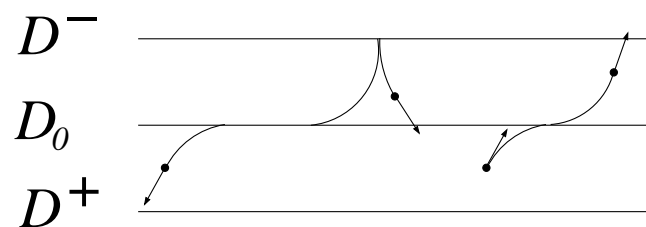
- if $\phi_2(t)$ vanishes on a nonempty interval $I \subset [t_i, t_f]$ from (1)

$$\psi_1 \cos(\theta(t)) + \psi_2 \sin(\theta(t)) + \psi_0 = 0$$

from lemma 2, ψ_1 and ψ_2 cannot be both zero then θ must remain constant on I

summarizing:

- type A trajectories are sequences of
 - arcs of circle (C) of radius 1 corresponding to regular control laws ($u_1 = \pm 1, u_2 = \pm 1$)
 - straight segments (S) corresponding to the singularity of u_2 ($u_1 = \pm 1, u_2 = 0$)
- straight line segments and points of inflection are on \mathcal{D}_0
- cusp tangents are perpendicular to \mathcal{D}^+ and \mathcal{D}^-
- **lemma** trajectories of type A and with no cusps are necessarily of one of the following kinds
 - C_a $0 \leq a \leq \pi$
 - $C_a C_b$ $0 < a \leq \frac{\pi}{2}, 0 < b \leq \frac{\pi}{2}$
 - $C_a S_d C_b$ $d > 0, 0 < a \leq \frac{\pi}{2}, 0 < b \leq \frac{\pi}{2}$



- to refine the large family of trajectories implied by type A a global geometric study would be needed

- a *boundary trajectory* is a trajectory $q : [t_i, t_f] \rightarrow \mathcal{M}$ such that $q(t_f)$ belongs to the boundary of the set of all reachable points from $q(t_i)$

PMP for boundary trajectories

if $q : [t_i, t_f] \rightarrow \mathcal{M}$ is a boundary trajectory, then it has a nontrivial adjoint vector $\psi(t)$ verifying (*) with $\psi_0 = 0$

type B trajectories

type B trajectories correspond to the singularity of the control component u_1 and their characterization requires geometric reasoning plus the application of PMP for boundary trajectories

lemma the search for optimal trajectories of type B can be restricted to the sufficient family of path types

$$l_a^+ l_b^- l_e^+ \text{ or } r_a^+ r_b^- r_e^+ \quad \text{with } 0 \leq a, b, e \leq \pi$$

in conclusion:

sufficient family of optimal trajectories for the unicycle (PMP + geometric reasoning)

(I)	$l_a^+ l_b^- l_e^+$ or $r_a^+ r_b^- r_e^+$	$0 \leq a \leq \pi, 0 \leq b \leq \pi, 0 \leq e \leq \pi$
(II)(III)	$C_a C_b C_e$ or $C_a C_b C_e$	$0 \leq a \leq b, 0 \leq e \leq b, 0 \leq b \leq \pi/2$
(IV)	$C_a C_b C_b C_e$	$0 \leq a \leq b, 0 \leq e \leq b, 0 < b \leq \pi/2$
(V)	$C_a C_b C_b C_e$	$0 \leq a \leq b, 0 \leq e \leq b, 0 < b \leq \pi/2$
(VI)	$C_a C_{\pi/2} S_d C_{\pi/2} C_b$	$0 \leq a \leq \pi/2, 0 \leq b < \pi/2, 0 \leq d$
(VII)(VIII)	$C_a C_{\pi/2} S_d C_b$ or $C_b S_l C_{\pi/2} C_a$	$0 \leq a \leq \pi, 0 \leq b \leq \pi/2, 0 \leq d$
(IX)	$C_a S_d C_b$	$0 \leq a \leq \pi/2, 0 \leq b \leq \pi/2, 0 \leq d$

- since $u_1 = \pm 1$ for all the path types contained in this family, they are admissible for the Reeds and Shepp's car; this implies that the family is also sufficient for the Reeds and Shepp's time-optimal control problem
- time-optimal trajectories for the Reeds and Shepp's car are paths of minimal length (recall that for Reeds and Shepp's car $u_1 = \pm 1$)

OPEN PROBLEMS

the techniques so far presented are fairly standard now, and the associated theoretical problems can be considered as solved

but: from an application viewpoint, many important issues deserve further research:

- **path planning in the presence of obstacles**: classical motion planning methods do not apply to WMRs because they ignore nonholonomic constraints
- **inclusion of dynamics**: for massive vehicles and/or at high speeds, consideration of robot dynamics is necessary for realistic control design
- **robust control design**: cope with disturbances and model perturbations (e.g., slipping)
- **use of exteroceptive feedback**: most control schemes require the measure of the WMR state; however, proprioceptive sensors, such as encoders, become unreliable in the long run \Rightarrow close the feedback loop with exteroceptive sensors providing absolute information about the robot localization in its workspace (e.g., vision)
- **WMRs not transformable in chained form**: such as a unicycle towing two or more trailers hitched at some distance from the midpoint of the previous wheel axle